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DYNAMIC STABILITY OF CYLINDRICAL PROPELLANT TANKS

by

Daniel D. Kana
Wen-Hwa Chu

FINAL REPORT, PART II

Contract No. NAS8-21282

Control No. DCN 1-8-75-00009 (IF)

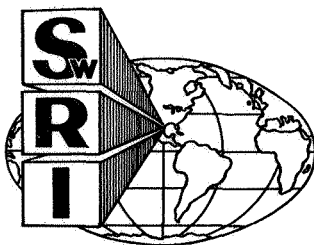
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Prepared for

**National Aeronautics and Space Administration
George C. Marshall Space Flight Center
Huntsville, Alabama**

2 June 1969



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**H. Norman Abramson, Director
Department of Mechanical Sciences**

PREFACE

This report constitutes the second of two volumes which summarize the work accomplished under Contract NAS8-21282. It contains supporting experimental data, a theoretical analysis, and a listing of a digital computer program designed for predicting dynamic stability of propellant tanks under longitudinal excitation. The first part of the work, which deals with the dynamic state of the system prior to instability, is summarized in Final Report, Part I, entitled "Influence of a Rigid Top Mass on the Response of a Pressurized Cylinder Containing Liquid."

Both Part I and Part II of this Final Report are published on the same date. They present significant extensions and refinements of concepts originated under previous investigations conducted for NASA-MSFC. Results of this preliminary work are summarized in "Dynamic Stability and Parametric Resonance in Cylindrical Propellant Tanks," by Daniel D. Kana, Wen-Hwa Chu, and Tom D. Dunham, Final Report, Contract No. NAS8-20329, SwRI Project No. 02-1876, January 17, 1968.

ABSTRACT

Dynamic instability and associated parametric resonance is a dominant form of response in a longitudinally excited cylindrical shell containing liquid. In order to assess the significance of such responses in a space vehicle propellant tank, the present paper is devoted to a theoretical and experimental study of their occurrence in a cylindrical shell system which includes the influences of axial preload, ullage pressure, partial liquid depth, and a finite top impedance. Donnell shell theory along with a modified Galerkin procedure is utilized to formulate equations which govern the stability of perturbations superimposed on an axisymmetric initial state of response. Stability boundaries are computed for a range of parameters affecting the region of principal parametric resonance and are compared with experimental results. It is found that liquid depth, top impedance, and ullage pressure have a strong influence on stability, while the effects of axial preload are relatively insignificant.

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NOMENCLATURE

a	radius of the shell
c_0	speed of sound in the liquid
c_s	E/ρ_s , speed of stress waves in the shell
E	modulus of elasticity
g	standard acceleration of gravity
H	h/a , nondimensional liquid depth
H_s	h_s/a , nondimensional thickness of shell
I_z	mass moment of inertia of top weight about z axis
l	length of the shell
m	one-half of the number of circumferential nodes; $\cos(m\theta)$
$N_{xxd}^*, N_{\theta\theta d}^*$	dynamic part of initial-state stress resultants [nondimensionalized by $(1 - \nu^2)/Eh_s$]
$N_{xxs}^*, N_{\theta\theta s}^*, N_{x\theta s}^*$	static part of initial-state stress resultants [nondimensionalized by $(1 - \nu^2)/Eh_s$]
n	axial wave number; $\sin n\pi x/l$
P_r	nondimensional pressure loading on shell, P_r/E
P_0, p_0	axial preload, ullage pressure
R, θ, X	cylindrical coordinates (space-fixed) nondimensionalized by radius a
U, V, W	shell displacements u, v, w , nondimensionalized by the radius a
X_0	nondimensional amplitude of axial excitation ($X_0 = \hat{x}_0/a$)
Z_0	top acceleration impedance (force/acceleration)
β	density parameter $\rho_l a/\rho_s h_s$

NOMENCLATURE (Cont'd)

ν	Poissons ratio
Φ	velocity potential, nondimensionalized by $\omega_0^2 a^2 / \omega_r$
ρ_l	mass density of liquid
ρ_s	mass density of the shell
τ	nondimensional time, $\tau = \omega_r t$
ω_0^2	liquid parameter c_0^2 / a^2
ω_r	response frequency
ω	excitation frequency
ω_k	natural frequency of m-k'th mode
Ω_i^2	designated frequency, nondimensionalized by a^2 / c_s^2
$\tilde{\Omega}_i^2$	designated frequency, nondimensionalized by $(1 - \nu^2) \times a^2 / c_s^2$
$\bar{\Omega}_i^2$	designated frequency, nondimensionalized by a^2 / c_0^2

Superscripts

(\sim)	the amplitude of ()
$(\dot{})$	$(d/d\tau)$ (), $\tau = \omega t$
$()^p$	related to initial state response

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INTRODUCTION

Dynamic instability and parametric resonance are known to occur in many engineering systems. Numerous classic examples have been studied in detail by Bolotin¹. In more recent experimental investigations², it was found that this type of behavior is dominant amidst a complex variety of responses which can be observed in a longitudinally excited model vehicle propellant tank which is not sufficiently reinforced with stiffeners. A theoretical and further experimental investigation³ was conducted for a longitudinally excited, liquid-filled cylindrical shell. It was found that the system initially tends to respond in a state comprised of linear axisymmetric modes. However, the resulting membrane stresses form a parametric load with respect to nonaxisymmetric perturbations superimposed on the initial state. Thus, for wide ranges of the excitation parameters, instability and subsequent parametric resonance results, and linear vibration theory is no longer adequate to predict the response of either liquid pressure or wall motion.

In order to assess the significance of such instabilities in a propellant tank which forms a component in an overall space vehicle structure, the present paper is devoted to a study of their occurrence in a cylindrical shell system which includes the influences of axial preload, ullage pressure, partial liquid depth, and a finite top impedance. A diagram of the system is shown in Figure 1, which includes the appropriate parameters and boundary conditions for both the initial and perturbed states. The initial state represents linear forced axisymmetric motion, whose responses have already

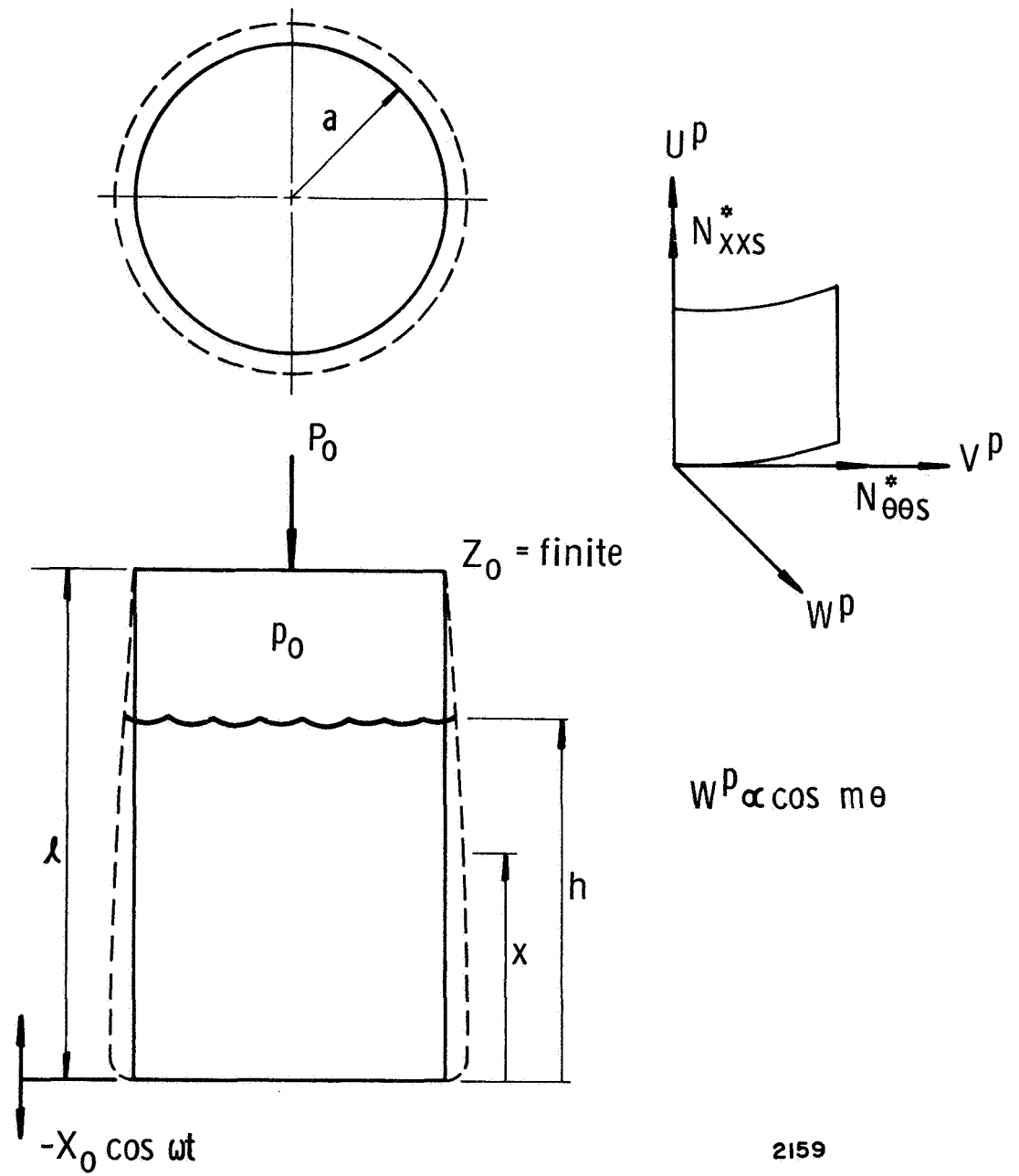
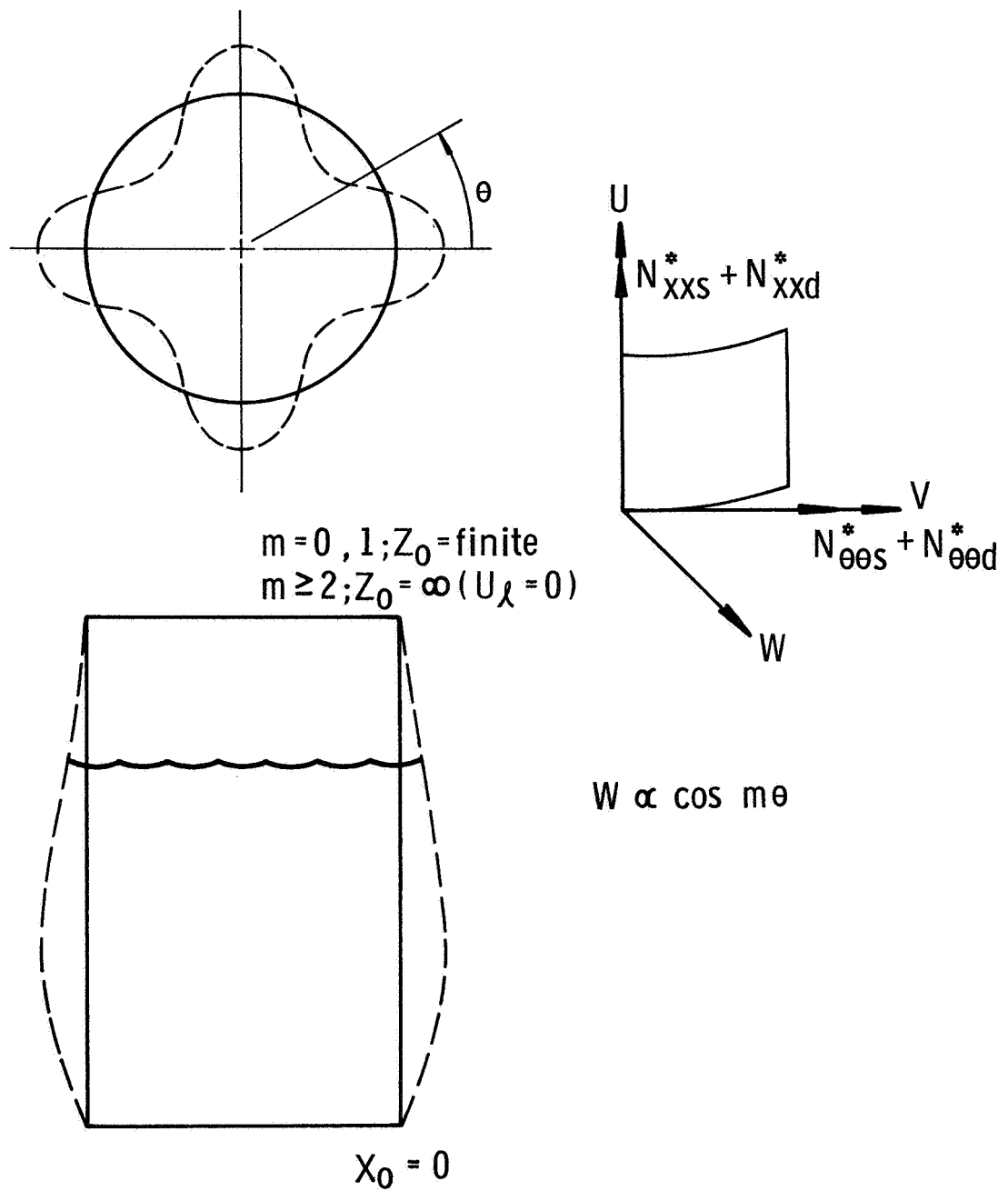


Figure 1a. Mechanism Of Dynamic Instability –
Initial State
 $m = 0$



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Figure 1b. Mechanism Of Dynamic Instability –
Perturbed State
 $m \geq 0$

been determined, along with natural frequencies and modal functions for the system⁴. Stability of motion in the perturbed state is the subject of the present paper, although results from Reference 4 must be utilized in part of the analysis. Note that, theoretically, the perturbed state can be either axisymmetric ($m = 0$) or nonaxisymmetric ($m > 0$); however, for a single tank system, the nonaxisymmetric form of instability is dominant.

DERIVATION OF STABILITY EQUATIONS

The perturbed motion represented by Figure 1b will be analyzed by means of Sander's nonlinear shell equations^{5,6} which are based on Donnell approximations. These equations contain nonlinear terms resulting from the rotation of shell elements as well as nonlinear strain-displacement relations. We will follow the philosophy of Bolotin¹ and assume that retaining only nonlinear terms which result from rotations is sufficient to determine dynamic stability. Compressible flow theory is used for the liquid. The motion is expanded into a series of the natural mode eigenvectors which were obtained from the solution of the free vibration problem⁴. A modified Galerkin procedure is then utilized to reduce the system to a linear second-order, time dependent set of coupled differential equations having periodic coefficients. The method is "modified" in the sense that the natural modal functions (finite series eigenvectors) are chosen as weighting functions, although they are not of closed form. An approximation of the perturbed motion will then be obtained by the use of only one eigenvector term of the series, so that the coupled set reduces to a single stability equation.

Thus, the governing shell equations are of the form

$$F_1 = L_{11}U + L_{12}V + L_{13}W - \tilde{\Omega}_r^2 \partial^2 U / \partial \tau^2 = 0 \quad (1a)$$

$$F_2 = L_{21}U + L_{22}V + L_{23}W - \tilde{\Omega}_r^2 \partial^2 V / \partial \tau^2 = 0 \quad (1b)$$

$$F_3 = L_{31}U + L_{32}V + L_{33}W - \tilde{\Omega}_r^2 \partial^2 W / \partial \tau^2 + \epsilon_d \tilde{L}_{33}W \cos \omega t + \frac{(1 - \nu^2)}{H_s} P_r = 0 \quad (1c)$$

where

$$\begin{aligned} L_{11} &= \frac{\partial^2}{\partial X^2} + \frac{1 - \nu}{2} \frac{\partial^2}{\partial \theta^2}, & L_{12} &= \frac{1 + \nu}{2} \frac{\partial^2}{\partial X \partial \theta} \\ L_{13} &= \nu \frac{\partial}{\partial X}, & L_{21} &= \frac{1 + \nu}{2} \frac{\partial^2}{\partial X \partial \theta} \\ L_{22} &= \frac{\partial^2}{\partial \theta^2} + \frac{1 - \nu}{2} \frac{\partial^2}{\partial X^2}, & L_{23} &= \frac{\partial}{\partial \theta} \\ L_{31} &= -\nu \frac{\partial}{\partial X}, & L_{32} &= -\frac{\partial}{\partial \theta} \end{aligned} \quad (2a)$$

$$\begin{aligned} L_{33} &= - \left[\frac{H_s^2}{12} \left(\frac{\partial^4}{\partial X^4} + 2 \frac{\partial^4}{\partial X^2 \partial \theta^2} + \frac{\partial^4}{\partial \theta^4} \right) + 1 \right] + \left[\frac{\partial}{\partial X} \left(N_{xxs}^* \frac{\partial}{\partial X} \right) \right. \\ &\quad \left. + \frac{\partial}{\partial X} \left(N_{x\theta s}^* \frac{\partial}{\partial \theta} \right) + \frac{\partial}{\partial \theta} \left(N_{\theta xs}^* \frac{\partial}{\partial X} \right) + \frac{\partial}{\partial \theta} \left(N_{\theta \theta s}^* \frac{\partial}{\partial \theta} \right) \right] \\ \tilde{L}_{33} &= \frac{\partial}{\partial X} \left(\hat{N}_{xxd}^* \frac{\partial}{\partial X} \right) + \frac{\partial}{\partial \theta} \left(\hat{N}_{\theta \theta d}^* \frac{\partial}{\partial \theta} \right) \end{aligned} \quad (2b)$$

$$\epsilon_d = 0 \text{ for free vibration}$$

$$\epsilon_d = 1 \text{ for forced vibration}$$

Pressure loading on the shell is given by:

$$\frac{(1 - \nu^2)}{H_s} P_r = -\tilde{\Omega}_0^2 \beta \frac{\partial \Phi}{\partial \tau} \text{ at } R = 1 \quad (3)$$

where the fluid velocity potential is governed by

$$\nabla^2 \Phi - \tilde{\Omega}_r^2 \partial^2 \Phi / \partial \tau^2 = 0 \quad (4)$$

Boundary conditions on the fluid and shell are:

At $X = 0$,

$$U = 0, \quad W = 0, \quad V = 0, \quad \partial^2 W / \partial X^2 = 0$$

At $X = l/a$,

$$W = 0, \quad V = 0, \quad \partial^2 W / \partial X^2 = 0$$

and

$$F_4 = \partial U / \partial X + Z^{**} \tilde{\Omega}_r^2 \partial^2 U / \partial \tau^2 = 0 \quad (5)$$

where

$$Z^{**} = \frac{Z_0(1 - \nu^2)}{2\pi\rho_s a^2 h_s} \quad \text{for } m = 0$$

$$Z^{**} = \frac{(1 - \nu^2)I_z}{4\rho_s h_s a^5} \quad \text{for } m = 1$$

$$Z^{**} = \infty \quad \text{for } m \geq 2$$

Solutions of the shell motion having a given circumferential displacement distribution will be sought as expansions of the m, k -th natural modes

$$U(\theta, \tau, X) = \cos m\theta \sum_{k=1}^K a_k(\tau) U_{mk}(X) = \sum_{k=1}^K a_k U_{1k} \quad (6a)$$

$$V(\theta, \tau, X) = \sin m\theta \sum_{k=1}^K a_k(\tau) V_{mk}(X) = \sum_{k=1}^K a_k U_{2k} \quad (6b)$$

$$W(\theta, \tau, X) = \cos m\theta \sum_{k=1}^K a_k(\tau) W_{mk}(X) = \sum_{k=1}^K a_k U_{3k} \quad (6c)$$

where for convenience we have defined a general shell displacement vector

$$\vec{U} = \vec{U}(U, V, W) = \vec{U}(U_1, U_2, U_3)$$

which is a function of both space and time and is associated with the fluid velocity potential Φ and upper shell displacement U_l .

The potential Φ satisfies Equation (4) which forms a constraint on the shell system. In order to interpret the fluid pressure loading as an apparent mass which is valid at the response frequency ω_r , we express the potential for forced motion as

$$\Phi(\theta, \tau, \omega_r, R, X) = \cos m\theta \sum_{k=1}^K \dot{a}_k(\tau) \Phi_{mk}(\omega_r, R, X) \quad (7)$$

Note that $\Phi_{mk}(\omega_r, R, X)$ is the component of Φ associated with a shell displacement component W_{mk} , and both liquid and shell motion is anticipated to be nearly periodic with responses at frequency ω_r . At the shell wall, we use the notation

$$\Phi_{mk}(\omega_r, l, X) = \Phi_{mk}(\omega_r, X)$$

For the special case of $\omega_r = \omega_k$, the system responds in the m, k -th natural mode, and the shell displacement modal functions form the vector

$$\vec{U}_k = \vec{U}_k(U_{1k}, U_{2k}, U_{3k})$$

This vector is a function of space only and is associated with the fluid velocity potential $\Phi_{mk}(\omega_k, X)$ and the top displacement U_{lk} . From the definition of natural frequencies, these modal functions satisfy

$$\sum_{j=1}^3 L_{ij} U_{jk} + \delta_{i3} \tilde{\Omega}_0^2 \Phi_{mk}(\omega_k, X) + \tilde{\Omega}_k^2 U_{ik} = 0 \quad (8a)$$

$$i = 1, 2, 3$$

$$\partial U_{lk} / \partial X = Z^{**} \tilde{\Omega}_k^2 U_{lk} \text{ at } X = l/a \quad (8b)$$

We now consider the forced motion. By means of a Galerkin procedure⁷, we form an expression for virtual work in the system

$$\sum_{i=1}^3 \int_S F_i(\vec{U}) \cdot U_{ik'} dS + \epsilon_m \int_C F_4(U_{lk}) \cdot U_{lk'} d\theta = 0 \quad (9)$$

where

$$\epsilon_m = 1 \text{ for } m = 0, 1$$

$$\epsilon_m = 0 \text{ for } m \geq 2$$

More specifically, we substitute Equations (3), (6) and (7) into Equations (1) and (5) and then by means of Equation (9) form an expression for virtual work between forces (expressed in terms of displacements) associated with the general forced motion and displacements associated with the m, k' -th natural mode. There results:

$$\begin{aligned}
& \sum_{k=1}^K \left[a_k \int_S \sum_{i=1}^3 \sum_{j=1}^3 U_{ik'} (L_{ij} U_{jk}) dS \right. \\
& - \ddot{a}_k \int_S \tilde{\Omega}_0^2 \beta_{mk}(\omega_r, X) U_{3k'} \cos m\theta dS - \tilde{\Omega}_r^2 \ddot{a}_k \int_S \sum_{i=1}^3 U_{ik'} U_{ik} dS \\
& + \epsilon_d a_k \cos \omega t \int_S U_{3k'} (\tilde{L}_{33} U_{3k}) dS + \epsilon_m \int_C \left(a_k \frac{\partial U_{1k}}{\partial X} U_{lk'} \right. \\
& \left. \left. + Z^{**} \tilde{\Omega}_r^2 \ddot{a}_k U_{lk} U_{lk'} \right) d\theta \right] = 0 \quad (10)
\end{aligned}$$

Upon use of Equations (8), reverting back to the more conventional displacement symbols in Equations (6) and carrying out the spatial integration, this can be written as

$$\begin{aligned}
& \sum_{k=1}^K [(K_{2k'k} - \epsilon_m K_{4k'k})(\tilde{\Omega}_r^2 \ddot{a}_k + \tilde{\Omega}_k^2 a_k) + K_{3k'k}(\tilde{\Omega}_r^2 \ddot{a}_k + \tilde{\Omega}_k^2 a_k)] \\
& + \sum_{k=1}^K \left\{ a_k \tilde{\Omega}_k^2 [M_{k'k}(\omega_k) + I_{k'k}] + \ddot{a}_k \tilde{\Omega}_r^2 [M_{k'k}(\omega_r) + I_{k'k}] \right. \\
& \left. + a_k N_{k'k} \cos \omega t \right\} = 0 \quad (11)
\end{aligned}$$

where $k' = 1, 2, 3, \dots, K$, and

$$K_{2k'k} = \int_0^{\ell/a} U_{mk'} U_{mk} dX \quad (12a)$$

$$K_{3k'k} = \int_0^{\ell/a} V_{mk'} V_{mk} dX \quad (12b)$$

$$K_{4k'k} = Z^{**} \int_0^{2\pi} U_{lk'} U_{lk} d\theta \quad (12c)$$

$$M_{k'k}(\omega_r) = \beta \frac{\omega_0^2}{\omega_r^2} \int_0^{\ell/a} W_{mk'} \Phi_{mk}(\omega_r, X) dX \quad (12d)$$

$$M_{k'k}(\omega_k) = \beta \frac{\omega_0^2}{\omega_k^2} \int_0^{\ell/a} W_{mk'} \Phi_{mk}(\omega_k, X) dX \quad (12e)$$

$$I_{k'k} = \int_0^{\ell/a} W_{mk'} W_{mk} dX \quad (12f)$$

$$N_{k'k} = - \int_0^{\ell/a} W_{mk'} \left(\hat{N}_{xxd}^* \frac{\partial^2 W_{mk}}{\partial X^2} + \frac{\partial \hat{N}_{xxd}^*}{\partial X} \frac{\partial W_{mk}}{\partial X} - m^2 \hat{N}_{\theta\theta d}^* W_{mk} \right) dX \quad (12g)$$

The coupled set of K equations (11) govern the perturbed motion, described in Figure 1b, for a given value of m. We will limit further discussion to the case of modes having $m \geq 2$. For these modes, the dominant motion is radial for the set of natural modes at lower frequencies. Thus, the terms under the first summation can be neglected, and, in matrix notation, Equations (11) become

$$\tilde{\Omega}_r^2[\tilde{M}] \{ \ddot{a} \} + \tilde{\Omega}_k^2[\tilde{K}] \{ a \} + [\tilde{T}] \{ a \} \cos \omega t = 0 \quad (13a)$$

where the elements of the k'-th row and k-th column of the corresponding matrices are

$$\tilde{M}_{k'k} = M_{k'k}(\omega_r) + I_{k'k}, \quad \tilde{K}_{k'k} = M_{k'k}(\omega_k) + I_{k'k}$$

$$\tilde{T}_{k'k} = N_{k'k}; \quad \text{and } \{a\} = \begin{Bmatrix} a_1 \\ a_2 \\ \cdot \\ \cdot \\ \cdot \\ a_K \end{Bmatrix}$$

Equation (13a) can further be written as

$$\{\ddot{a}\} + \frac{\tilde{\Omega}_k^2}{\tilde{\Omega}_r^2} [\underline{\tilde{M}}]^{-1} [\underline{\tilde{K}}] \{a\} + [\underline{\tilde{M}}]^{-1} [\underline{\tilde{T}}] \{a\} \cos \omega t = 0 \quad (13b)$$

When the flow is incompressible, $M_{k'k}$ is independent of ω and we have

$$[\underline{\tilde{M}}]^{-1} [\underline{\tilde{K}}] = [I]$$

where $[I]$ is the identity matrix. It must be emphasized that Equations (13) are not general differential equations in time but include the restriction of nearly periodic motion in the generalized apparent mass given by Equation (7).

EVALUATION OF MATRIX ELEMENTS

Modal Functions

Elements of the matrices in Equations (13) will now be evaluated from Equations (12d-g) in terms of the X -dependent natural modal functions (eigenvectors) of the system. These functions, which are not of closed form, have previously been determined from an eigenvalue problem⁴ in terms of the following series forms for the shell displacements:

$$U_{mk}(X) = \frac{1}{2} B_{2k} X^2 + B_{1k}(X - \ell/a) + B_{m0k} + \sum_{n=1}^N B_{mnk} \cos \lambda_n X \quad (14a)$$

$$V_{mk}(X) = \sum_{n=1}^N C_{mnk} \sin \lambda_n X \quad (14b)$$

$$W_{mk}(X) = \sum_{n=1}^N A_{mnk} \sin \lambda_n X \quad (14c)$$

and, for the velocity potential,

$$\Phi_{mk}(\omega_r, X) = \sum_{n=1}^N A_{mnk} \Psi_{mn}(\omega_r, X) \quad (15)$$

where $\Psi_{mn}(\omega_r, X)$ is a component function which satisfies Equation (4) for vibration at frequency ω_r , and corresponds to the $\sin \lambda_n X$ component function in W_{mk} through the boundary condition which must be satisfied at the tank wall⁴.

Mass Coefficients

The mass coefficient $I_{k'k}$ will now be developed from Equation (12f).

By means of Equation (14c), there results

$$\begin{aligned} I_{k'k} &= \int_0^{\ell/a} \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k'} \sin \lambda_{n'} X \sin \lambda_n X dX \\ &= \frac{\ell}{2a} \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k'} \delta_{n'n} = \frac{\ell}{2a} \sum_{n=1}^N A_{mnk} A_{mnk'} \end{aligned} \quad (16)$$

By substituting Equations (14c) and (15) into (12d), the liquid apparent mass coefficient corresponding to the response frequency ω_r becomes

$$M_{k'k}(\omega_r) = \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k} \mathcal{M}_{n'n}(\omega_r) \quad (17)$$

where

$$\mathcal{M}_{n'n}(\omega_r) = \beta \frac{\omega_0^2}{\omega_r^2} \int_0^{\ell/a} \Psi_{mn}(\omega_r, X) \sin \lambda_{n'} X dX$$

Except for a normalizing constant, the latter expression has also been evaluated in previous work. That is,

$$\mathcal{M}_{n'n}(\omega_r) = a_n^2 M_{mn'n} \quad (18)$$

where

$$a_n^2 = \int_0^{\ell/a} \sin^2 \lambda_{n'} X dX = \frac{\ell}{2a}$$

and $M_{mn'n}$ is given by Equation (18a) in Reference 4. Note, however, that the free index k used in the referenced expression is not the same k which is used to designate the natural mode herein, and we must also use $\omega_r = \omega$.

Finally, the apparent mass coefficient $M_{k'k}(\omega_k)$ given by Equation (12e) is obtained simply by substituting $\omega_r = \omega_k$ in Equations (17) and (18).

Parametric Coefficients

Upon substitution of Equation (14c) into (12g), we obtain:

$$N_{k'k} = \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k'} J_{n'n} \quad (19a)$$

where

$$J_{n'n} = \int_0^{\ell/a} \left(\lambda_n^2 \hat{N}_{xxd}^* \sin \lambda_n X - \lambda_n \frac{\partial \hat{N}_{xxd}^*}{\partial X} \cos \lambda_n X + m^2 \hat{N}_{\theta\theta d}^* \sin \lambda_n X \right) \sin \lambda_{n'} X dX \quad (19b)$$

The dynamic stress resultant amplitudes \hat{N}_{xxd}^* and $\hat{N}_{\theta\theta d}^*$ are produced by forced excitation in the axisymmetric initial state described in Figure 1a. These stress amplitudes can be expressed in terms of the amplitudes of the initial-state displacements by means of the usual stress-displacement equations

$$\hat{N}_{xxd}^* = \frac{\partial \hat{U}^P}{\partial X} + \nu \hat{W}^P \quad (20a)$$

$$\hat{N}_{\theta\theta d}^* = \hat{W}^P + \nu \frac{\partial \hat{U}^P}{\partial X} \quad (20b)$$

Thus, the parametric coefficients are partly determined by the initial state displacement amplitudes \hat{U}^P and \hat{W}^P .

The solution to the linear forced axisymmetric response of the initial state, in terms of displacements, has previously been given by Equation (25) in Reference 4. However, in the direct use of this equation, the appropriate elements of its matrices must include the substitution

$$M^{**} = Z^{**} \text{ for } m = 0, 1 \quad (21)$$

since an arbitrary acceleration impedance is allowed in the present problem, rather than only a rigid mass. Further, to allow for comparison of numerical and experimental data, it is convenient to express the initial-state displacements as ratios of the excitation amplitude X_0 . Therefore, the dynamic displacement amplitudes are of the form

$$\hat{U}^P = X_0 \left[\frac{1}{2} B_2^P X^2 + B_1^P \left(X - \frac{\ell}{a} \right) + B_{m0}^P + \sum_{n''=1}^N B_{mn''}^P \cos \lambda_{n''} X \right] \quad (22a)$$

$$\hat{W}^P = X_0 \sum_{n''=1}^N A_{mn''}^P \sin \lambda_{n''} X \quad (22b)$$

whose coefficients are completely determined by solving for the case of $X_0 = 1$.

The initial-state stresses can now be determined. Upon substituting Equations (22) into (20a), there results

$$\hat{N}_{xxd}^* = X_0 \left[B_1^P + B_2^P X - \sum_{n''=1}^N (\lambda_{n''} B_{mn''}^P - \nu A_{mn''}^P) \sin \lambda_{n''} X \right] \quad (23a)$$

and the derivative is

$$\frac{\partial \hat{N}_{xxd}^*}{\partial X} = X_0 \left[B_2^P - \sum_{n''=1}^N (\lambda_{n''} B_{mn''}^P - \nu A_{mn''}^P) \cos \lambda_{n''} X \right] \quad (23b)$$

Upon substituting Equations (22) into (20b), there results:

$$\hat{N}_{\theta\theta d}^* = X_0 \left[\nu(B_1^p + B_2^p X) + \sum_{n''=1}^N (A_{mn''}^p - \nu \lambda_{n''} B_{mn''}^p) \sin \lambda_{n''} X \right] \quad (23c)$$

For convenience of computation, these stress resultants and derivatives are expanded into complete Fourier series as follows:

$$\hat{N}_{xxd}^* = X_0 \sum_{n''=1}^N N_{1n''} \sin \lambda_{n''} X \quad (24a)$$

$$\hat{N}_{\theta\theta d}^* = X_0 \sum_{n''=1}^N N_{2n''} \sin \lambda_{n''} X \quad (24b)$$

$$\frac{\partial \hat{N}_{xxd}^*}{\partial X} = X_0 \left(B_2^p + \sum_{n''=1}^N N_{3n''} \cos \lambda_{n''} X \right) \quad (24c)$$

where

$$N_{1n''} = B_2^p \tilde{\chi}_{1n''} + B_1^p \tilde{\chi}_{0n''} - \lambda_{n''} B_{mn''}^p + \nu A_{mn''}^p$$

$$N_{2n''} = A_{mn''}^p + \nu (B_2^p \tilde{\chi}_{1n''} + B_1^p \tilde{\chi}_{0n''} - \lambda_{n''} B_{mn''}^p)$$

$$N_{3n''} = -\lambda_{n''}^2 B_{mn''}^p + \nu \lambda_{n''} A_{mn''}^p$$

and

$$\tilde{\chi}_{0n''} = \frac{2a}{\ell} \int_0^{\ell/a} \sin \lambda_{n''} X \, dX$$

$$\tilde{\chi}_{1n''} = \frac{2a}{\ell} \int_0^{\ell/a} X \sin \lambda_{n''} X \, dX$$

The parametric coefficients can now be completely evaluated.

Upon substitution of Equations (24) into Equations (19), there results

$$N_{k'k} = X_0 \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k'} \left[\sum_{n''=1}^N (\lambda_n^2 N_{1n''} d_{n''n'n} + m^2 N_{2n''} d_{n''n'n} - \lambda_n N_{3n''} e_{n''n'n}) - \lambda_n B_2^p e_{0n'n} \right] \quad (25)$$

where

$$e_{0n'n} = \int_0^{\ell/a} \cos \lambda_n X \sin \lambda_{n'} X dX$$

$$e_{n''n'n} = \int_0^{\ell/a} \cos \lambda_{n''} X \cos \lambda_n X \sin \lambda_{n'} X dX$$

$$d_{n''n'n} = \int_0^{\ell/a} \sin \lambda_{n''} X \sin \lambda_{n'} X \sin \lambda_n X dX$$

One-Term Approximation

For the m - k 'th natural mode in Equation (13a), set $k' = k$ to obtain

$$\tilde{\Omega}_r^2 \bar{M} \ddot{a}_k + \tilde{\Omega}_k^2 \bar{K} a_k + X_0 \bar{T} a_k \cos \omega t = 0 \quad (26)$$

where from Equations (16-19)

$$\bar{M} = \frac{\ell}{2a} \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k} M_{mn'n}(\omega_r) + \frac{\ell}{2a} \sum_{n=1}^N A_{mnk}^2$$

$$\bar{K} = \frac{\ell}{2a} \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k} M_{mn'n}(\omega_k) + \frac{\ell}{2a} \sum_{n=1}^N A_{mnk}^2$$

$$\overline{T} = \sum_{n=1}^N \sum_{n'=1}^N A_{mnk} A_{mn'k} \left[\sum_{n''=1}^N (\lambda_n^2 N_{1n''} d_{n''n'n} + m^2 N_{2n''} d_{n''n'n} - \lambda_n N_{3n''} e_{n''n'n}) - \lambda_n B_{20n'n}^p \right]$$

Equation (26) is a Mathieu equation whose stability properties are well known. To put it in a standard form⁸ for determining the stability boundaries for principal parametric (1/2-subharmonic) resonance, we set $\omega = 2\omega_r$ and obtain

$$\ddot{a}_k + (\bar{a} + 2\bar{q}X_0 \cos 2\tau)a_k = 0 \quad (27)$$

where

$$\bar{a} = \frac{\tilde{\Omega}_k^2 \overline{K}}{\tilde{\Omega}_r^2 \overline{M}}, \quad \bar{q} = \frac{\overline{T}}{2\tilde{\Omega}_r^2 \overline{M}}$$

The stability boundaries can then be approximated by

$$\begin{aligned} \bar{q}X_0 &= \bar{a} - 1 \text{ for } \bar{a} > 1 \\ \bar{q}X_0 &= 1 - \bar{a} \text{ for } \bar{a} < 1 \end{aligned} \quad (28)$$

In terms of input acceleration, which is convenient for experimental measurement, these become

$$g_x = X_0 \frac{\omega_a^2}{g} = \frac{\omega_a^2}{g} \left(\frac{|\bar{a} - 1|}{\bar{q}} \right) \quad (29)$$

THEORETICAL AND EXPERIMENTAL RESULTS

Experimental data for stability boundaries are obtained from the apparatus shown in Figure 2. All pertinent parameters, including the input

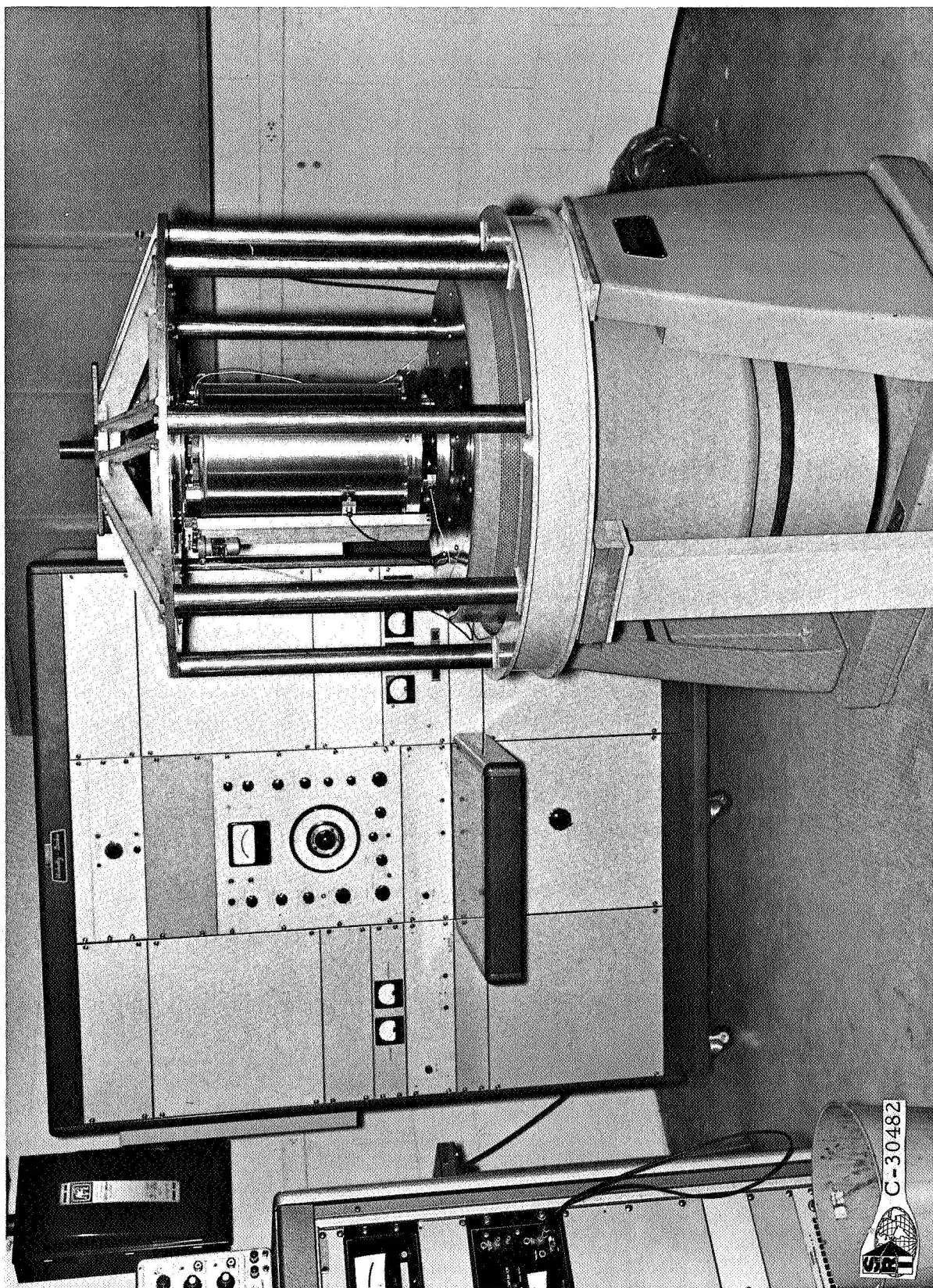
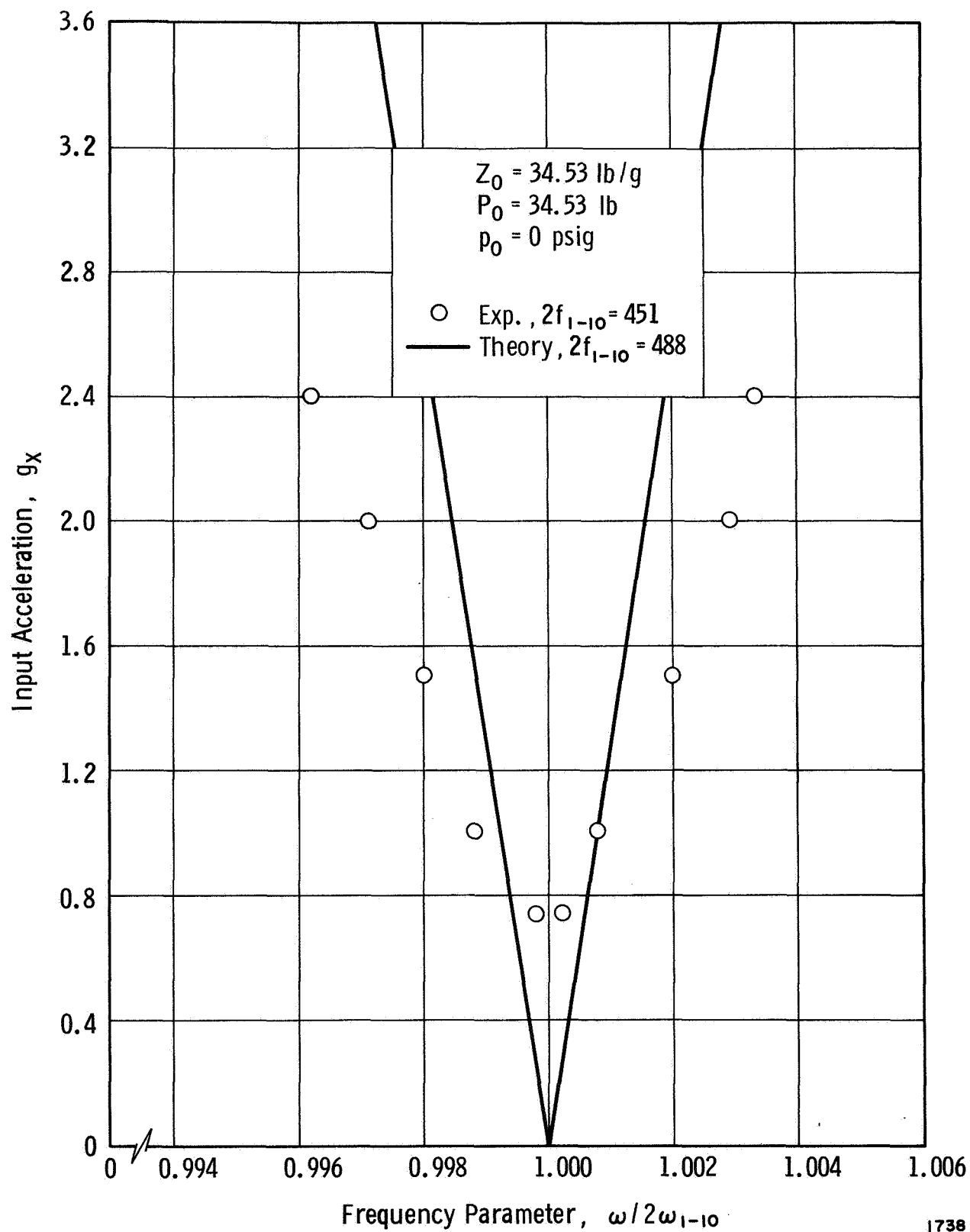


Figure 2. Experimental Apparatus

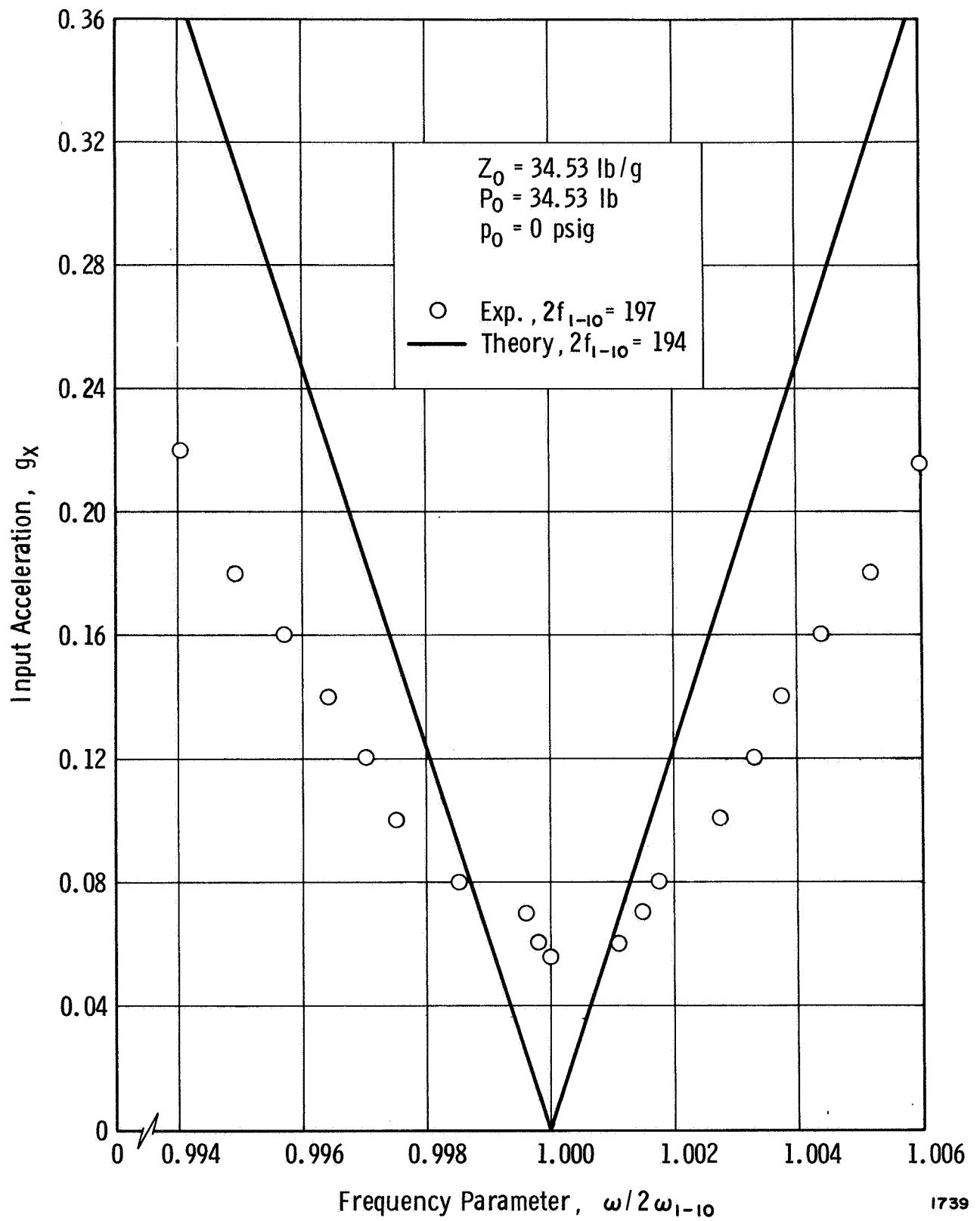
impedance Z_0 of the boundary condition at the tank top, could be measured. The use of acceleration impedance (force/acceleration) proved to be most convenient in this application. Variation of the impedance was achieved by using different rigid masses as well as the loading frame. The cylinder is made of 0.005-in. stainless steel, has a diameter of 10 in., and is 14.5 in. long (the same cylinder as that used in Reference 4).

Theoretical and experimental stability boundaries are compared in Figure 3 for the $k = 1$, $m = 10$ mode. Theoretical results are obtained from Equation (29) with $N = 5$ terms. Excitation conditions at or above the boundaries result in a principal parametric resonance whose mode shape is dominantly the $k = 1$, $m = 10$ natural mode, and whose frequency of motion is $1/2$ -subharmonic to the excitation. Experimental points were determined as the points of least acceleration where the parametric response would occur. It is apparent that significant deviation exists between theoretical and experimental results for the empty tank, and better agreement is achieved for greater liquid depths. After careful scrutiny, it was ascertained that the wider experimental stability boundaries are principally caused by imperfections in the cylinder. That is, split natural modes⁹ and spatially shifting modal patterns occurred so that one exact natural frequency did not exist. As a result, the experimental system shows a tendency to be more unstable than predicted by theory. This trend is apparent in all the data. It is possible that somewhat better agreement could be achieved by the use of some form of imperfection theory in the analysis. This possibility remains to be investigated.



a. $h/\lambda = 0$

Figure 3. Influence Of Liquid Depth On Stability



b. $h/\lambda = 0.414$

Figure 3. Influence Of Liquid Depth On Stability

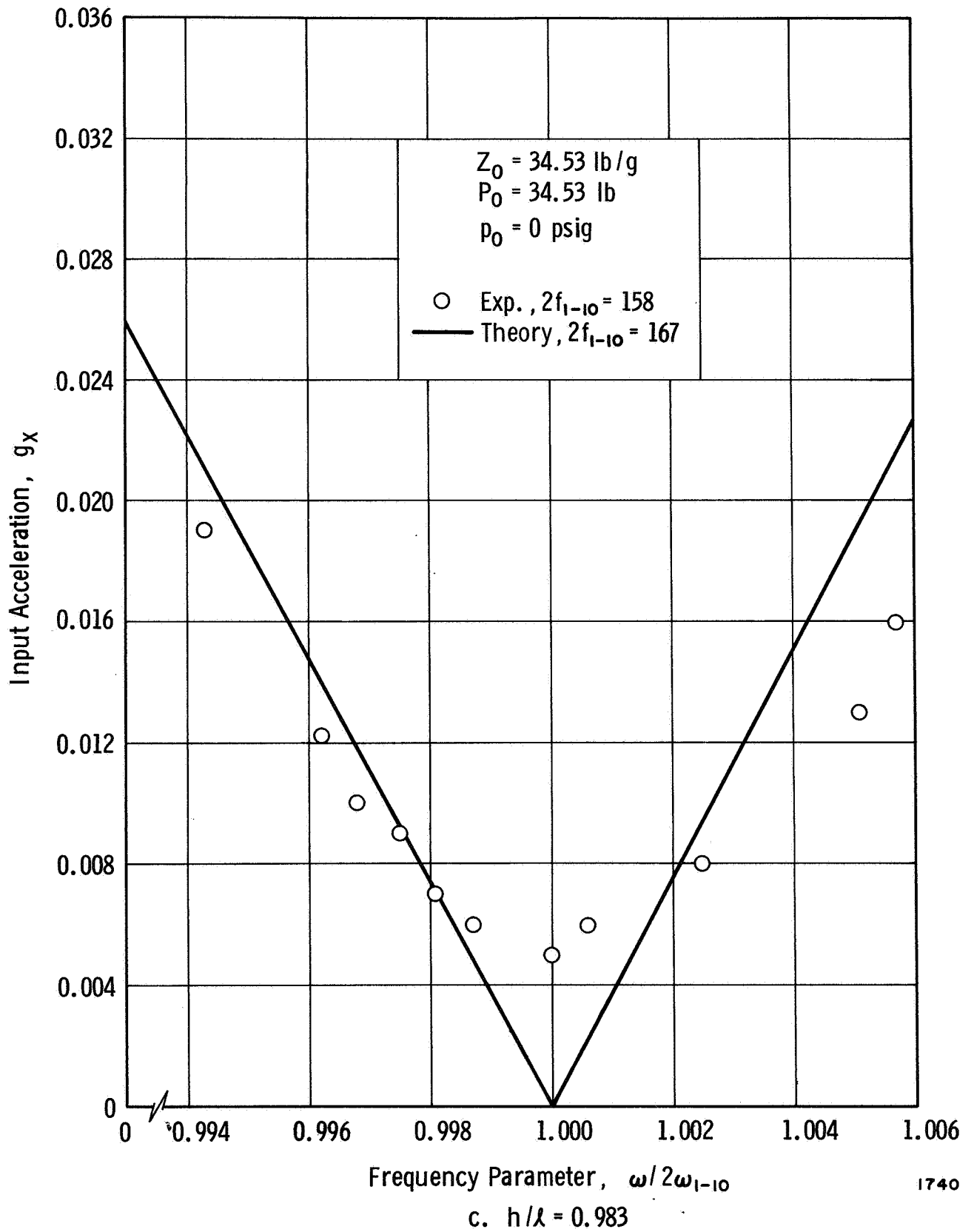


Figure 3. Influence Of Liquid Depth On Stability

It was desirable to determine the influence of the various system parameters on the stability boundaries for a given mode. This was done in terms of dimensional variables, in order to emphasize the complexity of this influence. For this purpose, it is necessary to understand the effects of the same parameters on the natural frequencies of the system. For convenience, some natural frequencies which were determined in the earlier work⁴ for several symmetric and one nonsymmetric mode are given as functions of liquid depth in Figure 4.

It is recognized that, in general, a more unstable system will possess a stability boundary whose acceleration ordinate is at a lower value for a given value of the frequency parameter $2\omega_{1-10}$. Therefore, in order to assess the effects of axial load, ullage pressure, liquid depth, and top impedance, a stability boundary acceleration g_{x1} was determined at an excitation frequency value of $\omega_{x1} = 0.996 (2\omega_{1-10})$ for a range of each of these parameters. Theoretical and experimental results are compared in Figures 5 through 8. These results must be compared with those in Figure 4 for proper interpretation. At a given liquid depth, increasing axial tension has only a small effect on natural frequencies and, likewise, only an insignificant effect on stability as shown in Figure 5. On the other hand, increasing ullage pressure significantly raises the natural frequencies of the nonsymmetric modes but leaves those of the lower symmetric modes essentially unchanged. Thus, as ω_{x1} approaches a natural frequency for a symmetric mode, the parametric excitation of the initial state is amplified, and the system becomes more unstable. This is reflected by the dips in the curves in

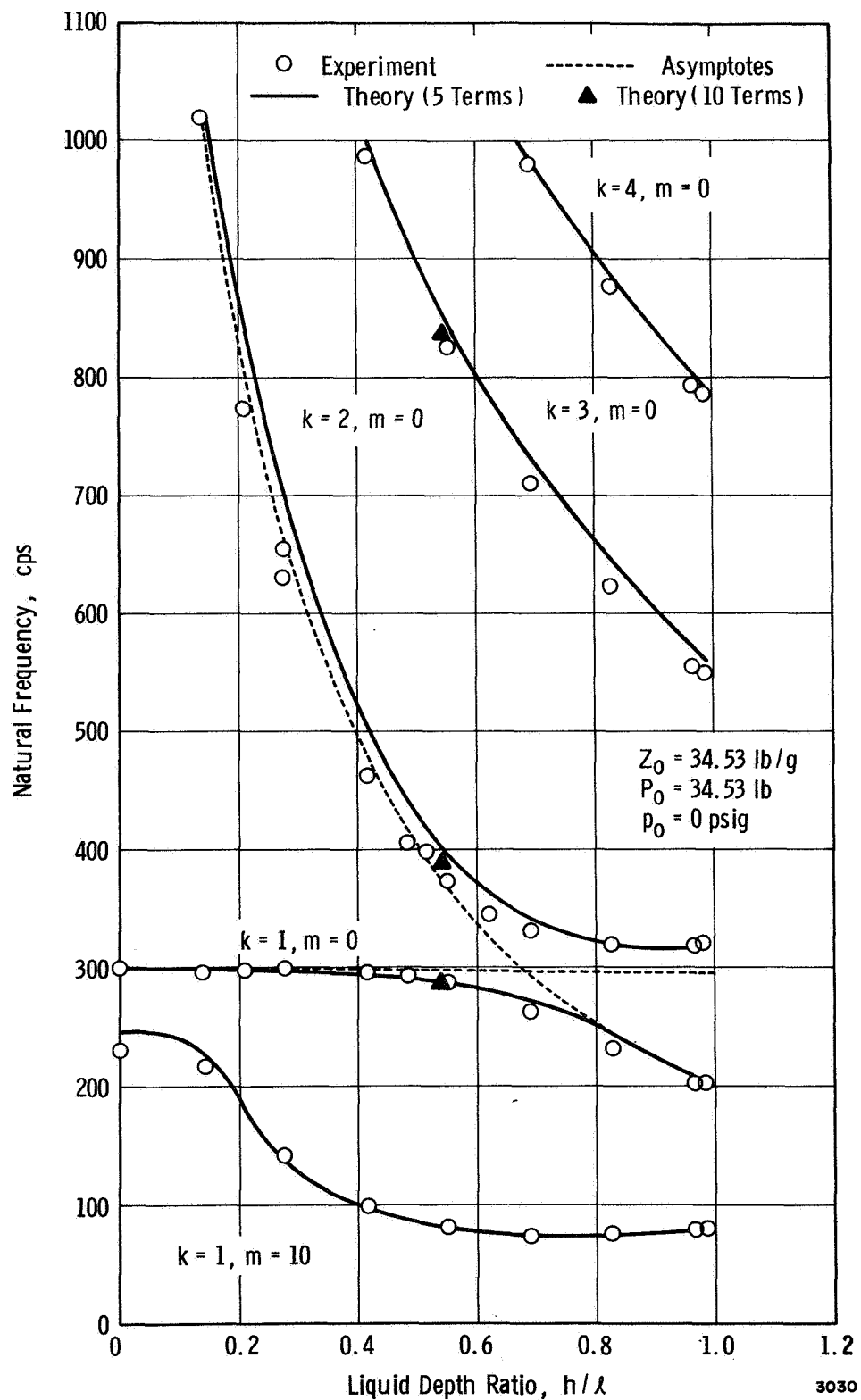


Figure 4. Natural Frequencies Of Partially Filled Tank

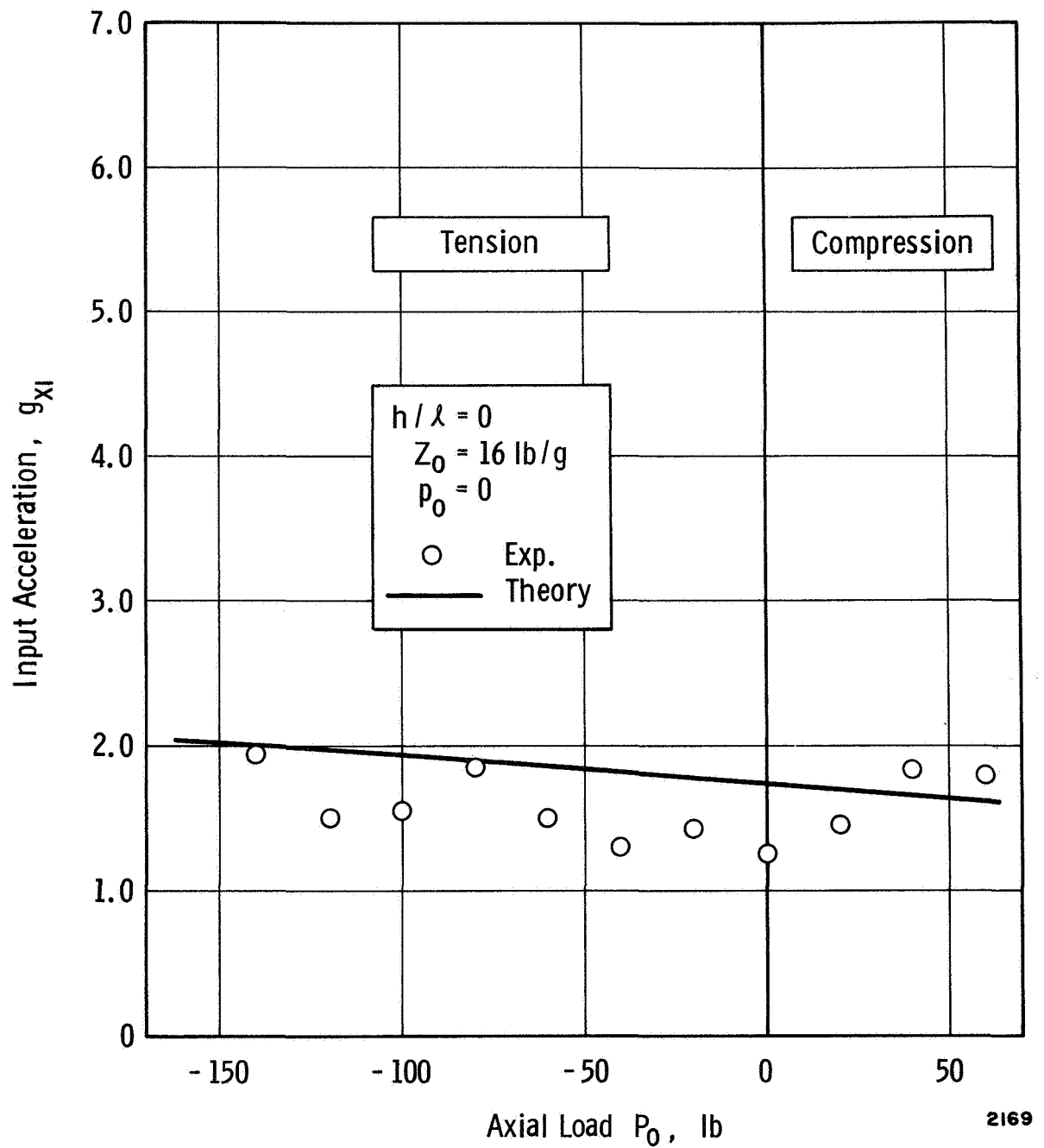


Figure 5. Influence Of Axial Load On Stability

Figure 6. It is also interesting to note that, at certain frequencies, the system becomes completely stable where the parametric coefficient in Equation (27) becomes zero.

Increasing liquid depth changes all natural frequencies, as shown in Figure 4, and has a profound influence on stability throughout the depth range, as shown in Figure 7. This results from the coincidence of ω_{x1} with natural frequencies of symmetric modes at certain points, as well as the provision of an increased distributed parametric loading on the tank wall.

The influence of top impedance on stability is shown in Figure 8. Increasing this impedance lowers the frequencies of symmetric modes while leaving the nonsymmetric mode frequencies unaltered. Thus, strong interaction can again be seen to occur. The dip in the curve occurs at an impedance such that ω_{x1} coincides with the natural frequency of the first symmetric mode.

It is obvious that variation of the above parameters can cause either an increase or decrease of stability, depending on the range of analysis. Further, it must be recognized that many nonsymmetric modes are present in the frequency range indicated in Figure 4, and each mode can become unstable as the one which was studied. Therefore, a complex pattern of instability and parametric resonance occurs with many overlapping regions of instability. The overall trend of the data shows good qualitative agreement between theory and experiment, although significant quantitative discrepancies exist because of the reasons previously discussed.

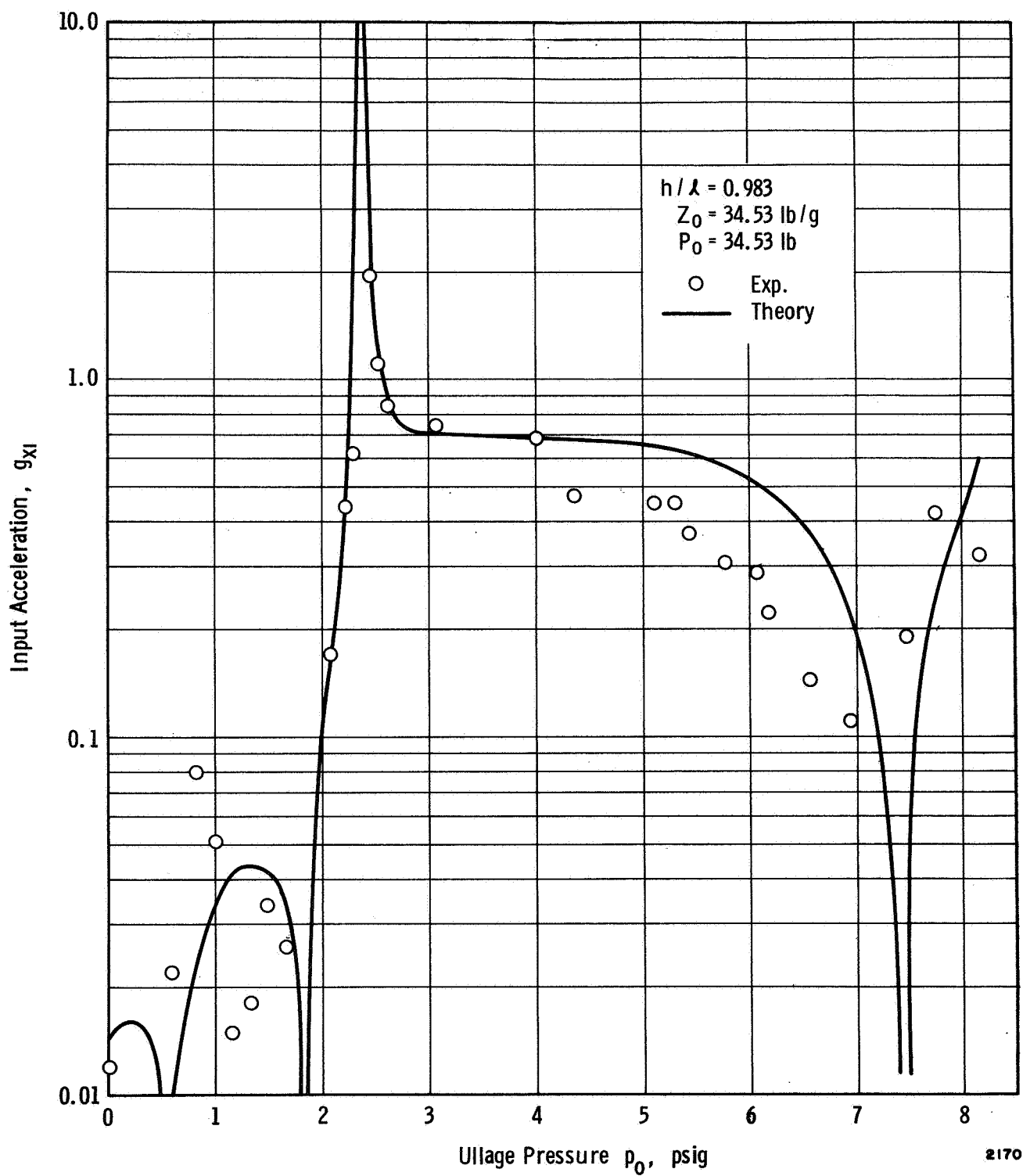


Figure 6. Influence Of Ullage Pressure On Stability

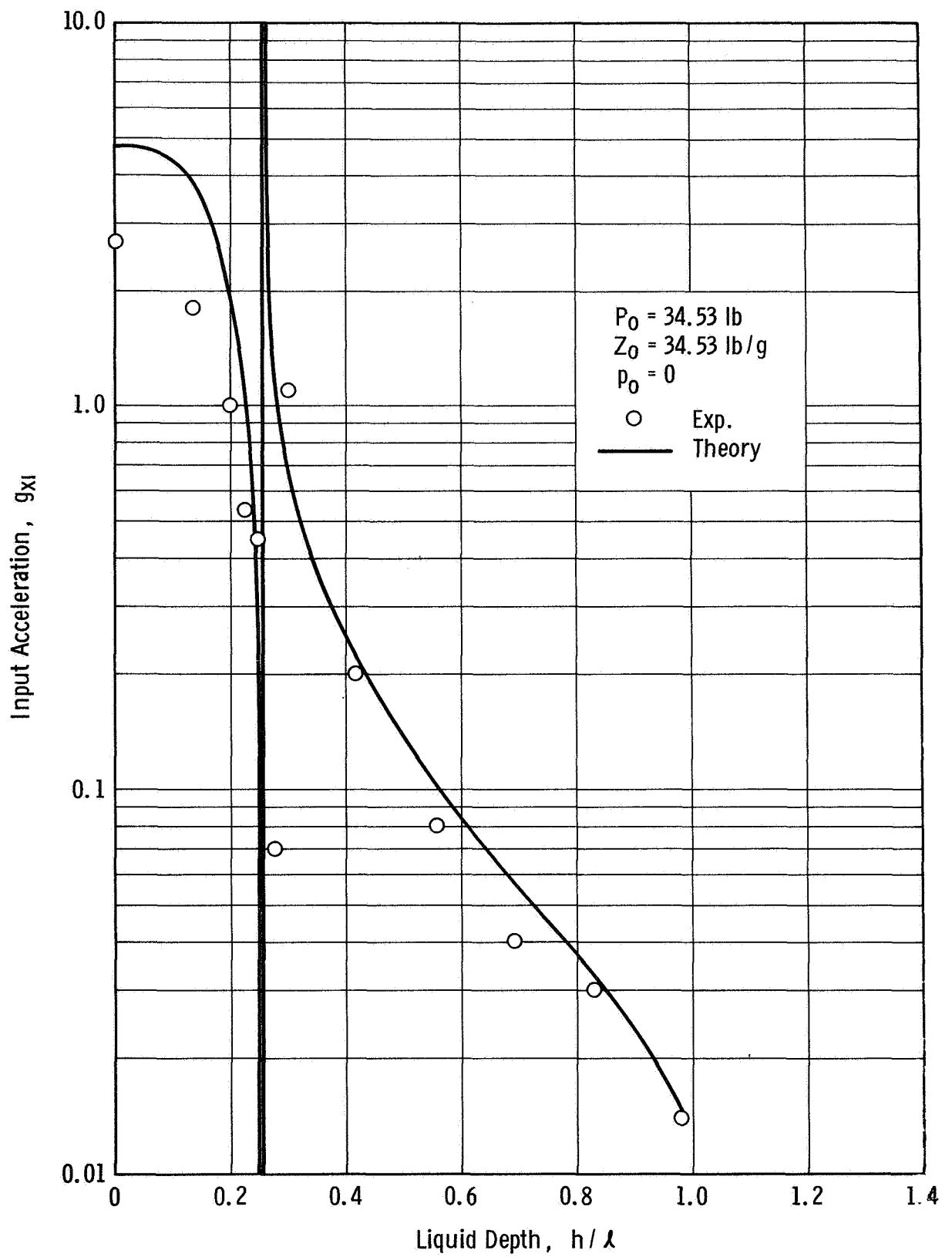


Figure 7. Influence Of Liquid Depth On Stability

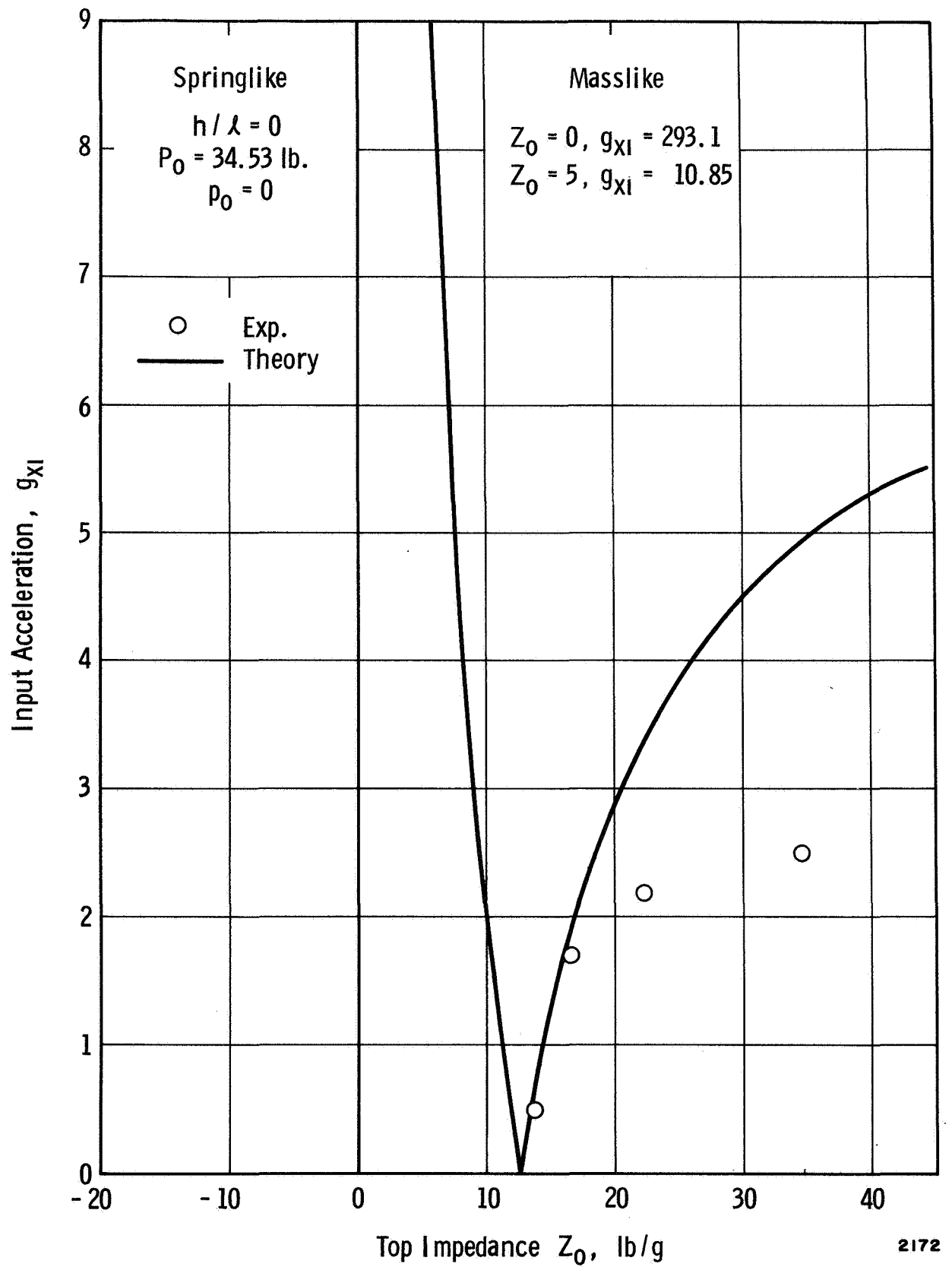


Figure 8. Influence Of Top Impedance On Stability

ACKNOWLEDGMENTS

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REFERENCES

1. Bolotin, V. V., The Dynamic Stability of Elastic Systems, Holden-Day, Inc., San Francisco, California (1964).
2. Kana, D. D. and Gormley, J. F., "Longitudinal Vibration of a Model Space Vehicle Propellant Tank," AIAA Journal of Spacecraft and Rockets, Vol. 4, No. 12, December 1967, pp. 1585-1591.
3. Kana, D. D. and Craig, R. R., Jr., "Parametric Oscillations of a Longitudinally Excited Cylindrical Shell Containing Liquid," AIAA Journal of Spacecraft and Rockets, Vol. 5, No. 1, January 1968, pp. 13-21.
4. Kana, D. D. and Chu, W. H., "Influence of a Rigid Top Mass on the Response of a Pressurized Cylinder Containing Liquid," Final Report Part I, Contract No. NAS8-21282, SwRI Project No. 02-2332, June 2, 1969, also AIAA Jour. Spacecraft and Rockets, 6, 2, February 1969, pp. 103-110.
5. Sanders, J. L., Jr., "Nonlinear Theories for Thin Shells," Q. Appl. Math., Vol. XXI, No. 1 (1963), pp. 21-36.
6. Sanders, J. L., Jr., "An Improved First Approximation Theory for Thin Shells," NASA Rept. 24, June 1959.
7. Duncan, W. J., "Galerkin's Method in Mechanics and Differential Equations," Aeronautical Research Committee Reports and Memoranda No. 1798, London (1937).
8. Cunningham, W. J., Introduction to Nonlinear Analysis, McGraw-Hill Book Co., Inc., New York, N. Y. (1958).
9. Tobias, S. A., "A Theory of Imperfection for the Vibration of Elastic Bodies of Revolution," Engineering, 172, pp. 409-411 (1951).

DIGITAL COMPUTER PROGRAM

INPUT DATA DESCRIPTION

Card No.	Fortran Symbol	Variable† Name	Units	Definition
1	RHO	386ρ	lb/in. ³	weight density of liquid
	RHOS	$386\rho_s$	lb/in. ³	weight density of the shell
	A0	a_0	in.	inner radius of the tank
	SH	h	in.	depth of liquid
	SHS	h_s	in.	thickness of shell
	SL	l	in.	length of the shell
2	P0	p_0	lb/in. ²	ullage pressure
	ENU	ν		Poissons ratio
	E	E	lb/in. ²	modulus of elasticity
	BMSTR	Z^{**}		nondimensional top impedance
	C0	c_0	in./sec ²	speed of sound in the liquid
3	NJ			no. of roots μ_{mj}
	N	N		no. terms in series expressions
4	W	$\omega/2\pi$	cps	excitation frequency
	BM	Z_0	lb sec ² /in.	top impedance
	CP0	P_0	lb	applied force
	NOPT			print option
	M	m		circumferential wave number
5	UMJ(I)	μ_{mj}		roots of $J'_m(\mu_{mj}) = 0, m \neq 0$
6	W	$\omega/2\pi$	cps	excitation frequency
	BM	Z_0	lb sec ² /in.	top impedance

†Note that some variables in computer program are slightly different from those as defined in NOMENCLATURE on pp. v to vi of text portion of this report.

Card No.	Fortran Symbol	Variable Name	Units	Definition
	CP0	P ₀	lb	applied force
	NOPT			print option
	M	m		circumferential wavenumber
7	UMJ(I)	μ_{mj}	m = 0	roots of $J'_m(\mu_{mj}) = 0$, m = 0

PROGRAM OUTPUT

Printed Output

1. Input data h , h_s , ℓ , a_0 , a , p_0 , Z^{**} , 386ρ , $386\rho_s$, E , c_0 , ν
2. In subroutine MITERS the mode no., eigenvalue, no. of iterations, no. of times Aitken's delta process is used, the eigenvector, and check eigenvalue and eigenvector
3. Mode no., $\tilde{\Omega}_k$, ω_k in rad/sec, and ω_k in cps
4. ω in cps, ω_k in cps, \overline{K} , \overline{T} , \overline{M} , \overline{a} , \overline{q} , $|X_0|$, g_x , and $\omega/2\omega_k$

PROGRAM NOTES

Subprograms Used

In addition to the main program the following subroutines are used:

1. BES, computes the Bessel functions J_n or I_n .
2. MATINV, computes the inverse of a real matrix.
3. MPRINT, prints matrix in matrix format.
4. MITERS, computes the eigenvalues and eigenvectors of a real or complex matrix by the power method.
5. NOPT is a print option that allows printing of intermediate results.

The following subroutines are included as a part of subprogram MITERS:

SWEEPX

NPNRMX

DPMLTX

SYMBOLIC LISTING

Some of the program FORTRAN symbols which were not defined in the Input Data Description are:

Fortran Symbol	Variable† Name	
A	a	
BMS	$M_s = 2\pi\rho_s a h_s \ell$	
HS	H_s	
CS	c_s	
H	H	
SLSTR	ℓ / a	
AONSQ	$q_{n'}^2$	
X10	x_{10}	
X20	x_{20}	
ALN(I)	λ_n	$n = 1, N$
ALNP(I)	$\lambda_{n'}$	$n' = 1, N$
X1(I)	$x_{1n'}$	$n' = 1, N$
X2(I)	$x_{2n'}$	$n' = 1, N$
X0B(I)	$\tilde{x}_{0n'}$	$n' = 1, N$
X1B(I)	$\tilde{x}_{1n'}$	$n' = 1, N$
ENB(I)	$\tilde{E}_{n'}$	$n' = 1, N$

†Corresponding to Final Report Part I.

Fortran Symbol	Variable Name	
ETA(J)	η_{mj}	$j = 1, NJ$
CJH(I)	C_{jH}	$j = 1, NJ$
CNJ(I, J)	$\tilde{C}_{n'j}$	$n' = 1, N; j = 1, NJ$
BMJN(I, J)	\tilde{B}_{mjn}	$j = 1, NJ; n = 1, N$
BN1(I)	N_{1mn}	$n = 1, N$
BN2(I)	N_{2mn}	$n = 1, N$
BN0(I)	N_{0mn}	$n = 1, N$
BOON(I)	\tilde{B}_{00n}	$n = 1, N$
BMMNN(I, J)	$M_{mn'n}$	$n' = 1, N; n = 1, N$
R1M(I, J)	$R_{1mn'n}$	
S2M(I, J)	$S_{2mn'n}$	
U1M(I, J)	$U_{1mn'n}$	
U2M(I, J)	$U_{2mn'n}$	
T3M(I, J)	$T_{3mn'n}$	
U3M(I, J)	$U_{3mn'n}$	
V2M(I, J)	$V_{2mn'n}$	
W1M(I, J)	$W_{1mn'n}$	
W2M(I, J)	$W_{2mn'n}$	
W3M(I, J)	$W_{3mn'n}$	
R2M(I, J)	$R_{2mn'n}$	
R3M(I, J)	$R_{3mn'n}$	
S1M(I, J)	$S_{1mn'n}$	

Fortran Symbol	Variable Name
S3M(I, J)	$S_{3mn'n}$
T1M(I, J)	$T_{1mn'n}$
T2M(I, J)	$T_{2mn'n}$
V1M(I, J)	$V_{1mn'n}$
X1M(I, 1)	$X_{1n'}$
U4M(1, I)	U_{4n}
V5M(1, I)	V_{5n}
R4M(1, I)	R_{4n}
R5M(1, I)	R_{5n}
R6M(1, I)	R_{6n}
O1M(I, 1)	$O_{1n'}$
P1M(I, 1)	$P_{1n'}$
Y1M(I, 1)	$Y_{1n'}$
Z1M(I, 1)	$Z_{1n'}$
Y2M(I, 1)	$Y_{2n'}$
Z2M(I, 1)	$Z_{2n'}$
Y3M(I, 1)	$Y_{3n'}$
Z3M(I, 1)	$Z_{3n'}$
P2M(I, 1)	$P_{2n'}$
Q2M(I, 1)	$Q_{2n'}$
V4M(1, I)	V_{4n}
S5M(1, I)	S_{5n}

Fortran Symbol	Variable Name
V3M(I, J)	$V_{3mn'n}$
V5M(1, I)	V_{5n}
S5M(1, I)	S_{5n}
UVW(I, J)	$[U]$
RST(I, J)	$[R]$
UTR(I, J)	$[U]^{-1}[R]$
UWR(I, J)	$[[U] - \tilde{\Omega}^2[R]]^{-1}$
AI(I)	$I_{0n'}$
QBH(I)	$\hat{Q}_{0n'}^B$
FH(I)	$F_{rn'}$
APH(I)	$[\{A_{mn}^P\} \{B_{mn}^P\} \{C_{mn}^P\} B_{mo}^P B_1^P B_2^P]'$
B0PH	B_0^P
B1PH	B_1^P
B2PH	B_2^P
CN1(I)	$N_{1n''}$
CN2(I)	$N_{2n''}$
CN3(I)	$N_{3n''}$
DNNN(I, J, K)	$d_{n''n'n}$
ENNN(I, J, K)	$e_{n''n'n}$
E0NN(I, J)	$e_{0n'n}$
TPNN(I, J)	$T'_{n'n_k}$
BMBAR	\overline{M}

Fortran Symbol	Variable Name
BKBAR	\overline{K}
BTBAR	\overline{T}
ABAR	\overline{a}
QBAR	\overline{q}
X0AQ	$ (1 - \overline{a})/\overline{q} $
X0STR	$ (1 - \overline{a})/\overline{q} \omega^2 a/g$
WSTR	$\omega/2\omega_k$
WKBSQ	$\tilde{\Omega}^2$
WKSQ	$\tilde{\Omega}_k^2$
AMNK(I, J)	$A_{mn'k}$

COMPUTATIONS LABORATORY		PROBLEM :		PAGE 2 OF 2	
SOUTHWEST RESEARCH INSTITUTE		PROGRAMMER :		DATE	
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C   SEPTEMBER 1967
C   MODIFIED 4 SEPTEMBER 1968
C   PROJECT 0 2 - 2 3 3 2
COMMON PI,H/COM1/A,A0,CSCOW/COM2/UMJ( 5)/COM3/AONSO/COM4/ALN(10)/
1  COM5/ALNP(10)
DIMENSION X1(10),X2(10),XOB(10),XIB(10), ENB(10),ETA( 5),
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2  BOON(10),BMMNN(10,10),R1M(10,10),R2M(10,10),R3M(10,10),
3  S1M(10,10),S2M(10,10),S3M(10,10),T1M(10,10),T2M(10,10),
4  T3M(10,10),U1M(10,10),U2M(10,10),U3M(10,10),V1M(10,10),
5  V2M(10,10),V3M(10,10),W1M(10,10),W2M(10,10),W3M(10,10),
6  UVW(33,33),RST(33,33),UWR(33,33),T(1000),UTRSV(33,36)
DIMENSION U4M(1,10),V4M(1,10),W4M(1,10),U5M(1,10),V5M(1,10),
1  W5M(1,10),U6M(1,10),V6M(1,10),W6M(1,10),X1M(10,1),Y1M(10,1),
2  Z1M(10,1),X2M(10,1),Y2M(10,1),Z2M(10,1),X3M(10,1),Y3M(10,1),
3  Z3M(10,1),R4M(1,10),S4M(1,10),T4M(1,10),R5M(1,10),S5M(1,10),
4  T5M(1,10),R6M(1,10),S6M(1,10),T6M(1,10),O1M(10,1),P1M(10,1),
5  Q1M(10,1),O2M(10,1),P2M(10,1),Q2M(10,1),O3M(10,1),P3M(10,1),
6  Q3M(10,1),UTR(33,36),BMMNNK(4,10,10),BMMNNO(4,10,10)
DIMENSION IROW(34),ICOL(33),AI(10),QBH(10),FH(33),APH(33)
DIMENSION WCPS(4),AMNK(33,4),CN1(10),CN2(10),CN3(10),
1  DNNN(10,10,10),ENNN(10,10,10),EONN(10,10),TPNN(10,10)
DIMENSION GUESS(33,1),VECTOR(33, 4),EIGVAL( 4),
1  NITER( 4),US(33,8),HH(33,29),NAKSR( 4),EIGCPS( 5)
PI = 3.14159265
G = 386.0
EPS = 1.0E-04
SMLST = 1.0E-07
NGUESS = 0
NMODE = 1
NITRSP = 100
EPSP = 0.5E-08
AITKEN = .9
NTAPE = 1
1000 READ 200, RHO,RHOS,A0,SH,SHS,SL
200 FORMAT (7F10.0)
IF (EOF,60)700,705
700 STOP
705 READ 205, P0,ENU,E,BMSTR,C0
205 FORMAT (2F10.0,3E12.3)
READ 215, NJ,N
215 FORMAT(3I5)
N3 = 3*N+3
NDIM = 33
RHO = RHO/386.
RHOS = RHOS/386.
A = A0+0.5*SHS
BMS = 2.0*PI*RHOS*A*SHS*SL
HS = SHS/A
CS = SQRTF(E/RHOS)
H = SH/A
SLSTR = SL/A
AONSO = 0.5*SLSTR
X10 = 0.5*SLSTR
X20 = SLSTR**2/3.0
PRINT 505
505 FORMAT (*1 PROJECT 0 2 - 2 3 3 2 *)
PRINT 530, SH,SHS,SL,A0,A,P0,BMSTR,RHO,RHOS,E,C0,ENU

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530 FORMAT(
      *0      H = *,E10.3,* (IN)*,17X,*HS = *,E10.3, SL006400
1* (IN)*,16X,*L = *,E10.3,* (IN)*//3X,*A0 = *,E10.3,* (IN)*,18X, SL006500
2*A = *,E10.3,* (IN)*//* P0 = *,E10.3,18H (LB-SEC**2/IN**4),3X, SL006600
36HM** = *,E10.3//* RHO = *,E10.3,18H (LB-SEC**2/IN**4),2X, SL006700
4 *RHOS = *,E10.3,18H (LB-SEC**2/IN**4)//4X, *E = *,E10.3, SL006800
5 11H (LB/IN**2),11X,*C0 = *,E10.3,* (IN/SEC)*,11X,*NU = *,E10.3) SL006900
      KOPT = 0 SL007000
2000 READ 210, W,BM,CPO,NOPT,M SL007100
210 FORMAT (3F10.0,2I5) SL007200
      READ 220, (UMJ(I),I=1,NJ) SL007300
220 FORMAT (5F15.0) SL007400
      BM = BM/386. SL007500
      IF (M)3,4,3 SL007600
4 BMSTR = (BM*(1.0-ENU**2))/(2.0*PI*RHOS*A**3*HS) SL007700
      JTER = 1 SL007900
      W = .996*(2.*WCPS(JTER)) SL008000
      WRAD = 2.*PI*W SL008100
      WKB = WCPS(JTER) SL008200
      WKSQ = (1.-ENU**2)*(2.*PI*WKB)**2*(A/CS)**2 SL008300
      GO TO 2 SL008400
3 CONTINUE SL008500
      DO 7000 ITER=1,1 SL008600
      IF (KOPT)6000,6005,6000 SL008700
6000 W = .996*( WCPS(ITER)) SL008800
      WKB = WCPS(ITER) SL008900
      WKSQ = (1.-ENU**2)*(2.*PI*WKB)**2*(A/CS)**2 SL009000
6005 WRAD = 2.*PI*W SL009100
2 OMEGA = (WRAD*A)/CS SL009200
      WSQ = OMEGA**2 SL009300
      WKBSQ = (1.0-ENU*ENU)*WSQ SL009400
      CSCOW = (CS/C0)*OMEGA SL009700
      DO 10 I=1,N SL009800
      ALN(I) = (I*PI*A)/SL SL009900
      ALNP(I) = (I*PI*A)/SL SL010000
      X1(I) = (2.0/(SLSTR*ALNP(I)**2))*(-1.0+(-1)**I) SL010100
      X2(I) = (4.0/ALN(I)**2)*(-1)**I SL010200
      X0B(I) = (2.0/(SLSTR*ALNP(I)))*(1.0-(-1)**I) SL010300
      X1B(I) = (2.0*(-1)**(I-1))/ALNP(I) SL010400
10 CONTINUE SL010500
      DO 5 I=1,N SL010600
      TM1 = ALNP(I)+CSCOW SL010700
      TM2 = ALNP(I)-CSCOW SL010800
      ENB(I) = (((1.0-COSF(TM1*H))/TM1)+((1.0-COSF(TM2*H))/TM2))/(2.0* SL010900
1 AONSQ) SL011000
5 CONTINUE SL011100
      DO 15 I=1,NJ SL011200
      ETA(I) = SQRTF(ABSF(UMJ(I)**2+CSCOW**2)) SL011300
      IF (UMJ(I)-CSCOW)20,25,25 SL011400
25 TM1 = EXPF(ETA(I)*H) SL011500
      CJH(I) = 0.5*(TM1+(1.0/TM1)) SL011600
      GO TO 15 SL011700
20 CJH(I) = COSF(ETA(I)*H) SL011800
15 CONTINUE SL011900
      DO 30 I=1,N SL012000
      TM1 = ALNP(I)*H SL012100
      SN = SINF(TM1) SL012200
      CN = COSF(TM1) SL012300
      DO 30 J=1,NJ SL012400
      IF (UMJ(J)-CSCOW)35,40,40 SL012500
40 TM2 = ETA(J)*H SL012600
      TM3 = EXPF(TM2) SL012700
      CSH = 0.5*(TM3+(1.0/TM3)) SL012800

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      SNH = 0.5*(TM3-(1.0/TM3))
      CNJ(I,J) = (1/(AONSO*(TM2**2+TM1**2)))*(TM1*CN*CSH-TM1-TM2*SN+SNH)
      GO TO 30
35  TM2 = ALNP(I)-ETA(J)
      TM3 = ALNP(I)+ETA(J)
      CNJ(I,J) = (0.5/AONSO)*(((1.0-COSF(TM3*H))/TM3)+((1.0-COSF(TM2*
1  H))/TM2))
30  CONTINUE
      DO 45 I=1,NJ
      DO 45 J=1,N
      SUMX = 0.
      SUMY = 1.0E-50
      K = 0
60  TRM = ((-1)**K*DKN(K,J)*EMKJ(M,K,I))/CJH(I)
      SUMX = SUMX+TRM
      IF (ABSF((SUMX-SUMY)/SUMY)-EPS)50,50,55
55  SUMY = SUMX
      K = K+1
      IF (K-100)60,50,50
50  RMJN(I,J) = SUMY
45  CONTINUE
      DO 65 I=1,N
      TM1 = (1.0-ENU+ENU)*WSQ
      TM2 = (A/SL)*0.5*BMSTR*WSQ
      TM3 = -(A/SL)*(-1)**I*BMSTR*WSQ
      TM4 = -(A/SL)*BMSTR*WSQ
      TM5 = 1.0-(A/SL)*X10
      TM6 = 1.0*(M**2/(2.0*(1.0+ENU)*WSQ))
      TM7 = 0.5*(1.0-ENU)*M*M*X20
      BN0(I) = -((A/SL)**2*((2.0+TM1*X20-TM7)/(2.0*TM1))*(TM3/TM2)-TM5*
1  TM6)/(1.0+((2.0+TM1*X20-TM7)/(2.0*TM1))*(TM4/TM2)*(A/SL)**2-
2  TM5*TM6)
      BN1(I) = +(A/SL)*(1.0+BN0(I))
      BN2(I) = ((TM4*BN0(I)+TM3)*(A/SL)**2)/TM2
65  CONTINUE
      DO 66 I=1,N
      SUMX = 0.
      SUMY = 1.0E-50
      K = 1
69  TRM = (-1)**K*DKN(K,I)*EMKJ(M,K,0)
      SUMX = SUMX+TRM
      IF (ABSF((SUMX-SUMY)/SUMY)-EPS)67,67,68
68  SUMY = SUMX
      K = K+1
      IF (K-100)69,67,67
67  BOON(I) = SUMY/COSF(CSCOW*H)
66  CONTINUE
      DO 70 I=1,N
      DO 70 J=1,N
      SUMX = 0.
      SUMY = 1.0E-50
      IF (M)86,86,87
86  K = 1
      GO TO 85
87  K=0
85  TRM = RMK(M,K)*DKN(K,J)*DKNB(K,I)
      SUMX = SUMX+TRM
      IF (ABSF((SUMX-SUMY)/SUMY)-EPS)75,75,80
80  SUMY = SUMX
      K = K+1
      IF (K-100)85,85,75
75  SUMZ = 0.

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DO 90 L=1,NJ
CALL BES(M,UMJ(L),0,XJ0,T)
TRM = XJ0*BMMN(I,J)*CNJ(I,L)
SUMZ = SUMZ+TRM
90 CONTINUE
IF (M)91,92,91
92 CALL BES(0,CSC0W,0,XJ0,T)
CALL BES(1,CSC0W,0,XJ1,T)
TRM = -BOON(J)*ENB(I)+(((2.0*ENB(I))/(CSC0W**2+COSF(CSC0W*H)))-
1 ((XJ0*DKNB(0,I))/(CSC0W*XJ1)))*DKN(0,J)
BMMNN(I,J) = ((RHO*A)/(RHOS*SHS))*(SUMY-SUMZ+TRM)
GO TO 70
91 TRM = 0,
BMMNN(I,J) = ((RHO*A)/(RHOS*SHS))*(SUMY-SUMZ+TRM)
IF (ITER)93,70,93
93 CONTINUE
IF (KOPT)6010,6015,6010
6010 BMMNN0(ITER,I,J) = BMMNN(I,J)
GO TO 70
6015 BMMNNK(ITER,I,J) = BMMNN(I,J)
70 CONTINUE
IF (M)6025,6020,6025
6025 CONTINUE
IF (KOPT)2000,6020,2000
6020 CONTINUE
DO 95 I=1,N
DO 95 J=1,N
IF (I-J)100,105,100
105 R1M(I,J) = 1.0+BMMNN(I,J)
GO TO 95
100 R1M(I,J) = BMMNN(I,J)
95 CONTINUE
DO 110 I=1,N
DO 110 J=1,N
IF (I-J)115,120,115
120 S2M(I,J) = 1,
GO TO 110
115 S2M(I,J) = 0,
110 CONTINUE
DO 125 I=1,N
DO 125 J=1,N
TM1 = 1.0+(HS*HS*(ALN(J)**2+M*M)**2)/12,
TM2 = (P0/E)*(0.5*ALN(J)**2+M*M)
IF (I-J)130,135,130
135 X3N = (A/SL)*(0.5*H**2-((1.-COS(2.*ALN(J)*H))/(4.*ALN(J)**2)))
TM3 = X3N*M*M*(RHO*G*A/E)
U1M(I,J) = TM1+(TM2+TM3-((CP0 *ALN(J)**2)/(2.0*PI*A*A*E)))*((1.0-
1 ENU**2)/HS)
GO TO 125
130 TM3 = ALN(J)-ALNP(I)
TM4 = ALN(J)+ALNP(I)
X3N = (A/SL)*(((1.-COS(TM3*H))/TM3**2)-((1.-COS(TM4*H))/TM4**2))
U1M(I,J) = (X3N*M*M)*((1.0-ENU**2)/HS)*(RHO*G*A/E)
125 CONTINUE
DO 140 I=1,N
DO 140 J=1,N
IF (I-J)145,150,145
150 U2M(I,J) = -ENU*ALN(J)
T3M(I,J) = 1.0
U3M(I,J) = M
V2M(I,J) = ALN(J)**2+0.5*(1.0-ENU)*M*M
W1M(I,J) = M

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SL019100
SL019200
SL019300
SL019400
SL019500
SL019600
SL019700
SL019800
SL019900
SL020000
SL020100
SL020200
SL020300
SL020400
SL020500
SL020600
SL020700
SL020800
SL020900
SL021000
SL021100
SL021200
SL021300
SL021400
SL021500
SL021600
SL021700
SL021800
SL021900
SL022000
SL022100
SL022200
SL022300
SL022400
SL022500
SL022600
SL022700
SL022800
SL022900
SL023000
SL023100
SL023200
SL023300
SL023400
SL023500
SL023600
SL023700
SL023800
SL023900
SL024000
SL024100
SL024200
SL024300
SL024400
SL024500
SL024600
SL024700
SL024800
SL024900
SL025000
SL025100
SL025200

W2M(I,J) = -0.5*(1.0+ENU)*M*ALN(J)	SL025300
W3M(I,J) = 0.5*(1.0-ENU)*ALN(J)**2+M*M	SL025400
GO TO 140	SL025500
145 U2M(I,J) = 0,	SL025600
T3M(I,J) = 0,	SL025700
U3M(I,J) = 0,	SL025800
V2M(I,J) = 0,	SL025900
W1M(I,J) = 0,	SL026000
W2M(I,J) = 0,	SL026100
W3M(I,J) = 0,	SL026200
140 CONTINUE	SL026300
DO 155 I=1,N	SL026400
DO 155 J=1,N	SL026500
R2M(I,J) = 0,	SL026600
R3M(I,J) = 0,	SL026700
S1M(I,J) = 0,	SL026800
S3M(I,J) = 0,	SL026900
T1M(I,J) = 0,	SL027000
T2M(I,J) = 0,	SL027100
155 CONTINUE	SL027200
DO 160 I=1,N	SL027300
DO 160 J=1,N	SL027400
IF (I-J)161,162,161	SL027500
162 V1M(I,J) = -ALN(J)*ENU	SL027600
GO TO 160	SL027700
161 V1M(I,J) = 0,	SL027800
160 CONTINUE	SL027900
DO 165 I=1,N	SL028000
X1M(I,1) = X2M(I,1) = X3M(I,1) = 0.	SL028100
U4M(I,1) = U5M(I,1) = U6M(I,1) = 0.	SL028200
V5M(I,1) = W5M(I,1) = V6M(I,1) = W6M(I,1) = W4M(I,1) = 0.	SL028300
R4M(I,1) = S4M(I,1) = T4M(I,1) = 0.	SL028400
R5M(I,1) = T5M(I,1) = 0.	SL028500
R6M(I,1) = S6M(I,1) = T6M(I,1) = 0.	SL028600
Q1M(I,1) = Q2M(I,1) = Q3M(I,1) = 0.	SL028700
P1M(I,1) = P3M(I,1) = Q1M(I,1) = Q3M(I,1) = 0.	SL028800
Y1M(I,1) = ENU*X0R(I)	SL028900
Z1M(I,1) = ENU*X1R(I)	SL029000
Y2M(I,1) = M**2*(.5*(1.-ENU))*X1(I)	SL029100
Z2M(I,1) = M**2*(.25*(1.-ENU))*X2(I)	SL029200
Y3M(I,1) = M*(.5*(1.+ENU))*X0R(I)	SL029300
Z3M(I,1) = M*(.5*(1.+ENU))*X1R(I)	SL029400
P2M(I,1) = X1(I)	SL029500
Q2M(I,1) = .5*X2(I)	SL029600
V4M(I,1) = -1.	SL029700
S5M(I,1) = (-1)**I/(1.-ENU**2)	SL029800
DO 165 J=1,N	SL029900
IF (I-J)170,175,170	SL030000
175 V3M(I,J) = -ALN(J) *0.5*M*(1.0+ENU)	SL030100
GO TO 165	SL030200
170 V3M(I,J) = 0,	SL030300
165 CONTINUE	SL030400
X5M = Z4M = O4M = P4M = Q4M = P5M = 0.	SL030500
X4M = -1.	SL030600
Y4M = SL/A	SL030700
Y5M = 1./BMSTR	SL030800
Z5M = Y4M*Y5M	SL030900
O5M = 1./(1.-ENU**2)	SL031000
Q5M = (.5*(SL/A)**2)/(1.-ENU**2)	SL031100
Y6M = M**2*(.5*(1.-ENU))**(-((SL/A)*X10)	SL031200
Z6M = -(1.-.5*(M**2*(1.-ENU))*X20)	SL031300
X6M = .5*M**2*(1.-ENU)	SL031400

Q6M = 1,	SL031500
P6M = X10-SL/A	SL031600
Q6M = .5*X20	SL031700
IF (M)166,166,167	SL031800
167 X5M = 1,	SL031900
Y5M = 0,	SL032000
Z5M = .5*(SL/A)+.2	SL032100
Q5M = 0,	SL032200
P5M = 0,	SL032300
Q5M = 0,	SL032400
DO 168 I=1,N	SL032500
V5M(I,I) = (-1)+I	SL032600
S5M(I,I) = 0.	SL032700
168 CONTINUE	SL032800
166 DO 185 I=1,N	SL032900
JK = I+N	SL033000
JL = I+2*N	SL033100
IJ = 3*N+1	SL033200
IK = 3*N+2	SL033300
IL = 3*N+3	SL033400
UVW(I,IJ) = X1M(I,1)	SL033500
UVW(I,IK) = Y1M(I,1)	SL033600
UVW(I,IL) = Z1M(I,1)	SL033700
UVW(JK,IJ) = X2M(I,1)	SL033800
UVW(JK,IK) = Y2M(I,1)	SL033900
UVW(JK,IL) = Z2M(I,1)	SL034000
UVW(JL,IJ) = X3M(I,1)	SL034100
UVW(JL,IK) = Y3M(I,1)	SL034200
UVW(JL,IL) = Z3M(I,1)	SL034300
UVW(IJ,I) = U4M(1,I)	SL034400
UVW(IJ,JK) = V4M(1,I)	SL034500
UVW(IJ,JL) = W4M(1,I)	SL034600
UVW(IK,I) = U5M(1,I)	SL034700
UVW(IK,JK) = V5M(1,I)	SL034800
UVW(IK,JL) = W5M(1,I)	SL034900
UVW(IL,I) = U6M(1,I)	SL035000
UVW(IL,JK) = V6M(1,I)	SL035100
UVW(IL,JL) = W6M(1,I)	SL035200
DO 185 J=1,N	SL035300
IK = J+N	SL035400
IL = J+2*N	SL035500
UVW(I,J) = U1M(I,J)	SL035600
UVW(I,JK) = V1M(I,J)	SL035700
UVW(I,IL) = W1M(I,J)	SL035800
UVW(JK,J) = U2M(I,J)	SL035900
UVW(JK,IK) = V2M(I,J)	SL036000
UVW(JK,IL) = W2M(I,J)	SL036100
UVW(JL,J) = U3M(I,J)	SL036200
UVW(JL,IK) = V3M(I,J)	SL036300
UVW(JL,IL) = W3M(I,J)	SL036400
185 CONTINUE	SL036500
DO 190 I=1,N	SL036600
JK = I+N	SL036700
JL = I+2*N	SL036800
IJ = 3*N+1	SL036900
IK = 3*N+2	SL037000
IL = 3*N+3	SL037100
RST(I,IJ) = O1M(I,1)	SL037200
RST(I,IK) = P1M(I,1)	SL037300
RST(I,IL) = Q1M(I,1)	SL037400
RST(JK,IJ) = O2M(I,1)	SL037500
RST(JK,IK) = P2M(I,1)	SL037600

RST(JK,IL) = Q2M(1,1)	SL037700
RST(JL,IJ) = Q3M(1,1)	SL037800
RST(JL,JK) = P3M(1,1)	SL037900
RST(JL,IL) = Q3M(1,1)	SL038000
RST(IJ,I) = R4M(1,1)	SL038100
RST(IJ,JK) = S4M(1,1)	SL038200
RST(IJ,JL) = T4M(1,1)	SL038300
RST(IK,I) = R5M(1,1)	SL038400
RST(IK,JK) = S5M(1,1)	SL038500
RST(IK,JL) = T5M(1,1)	SL038600
RST(IL,I) = R6M(1,1)	SL038700
RST(IL,JK) = S6M(1,1)	SL038800
RST(IL,JL) = T6M(1,1)	SL038900
DO 190 J=1,N	SL039000
IK = J+N	SL039100
IL = J+2*N	SL039200
RST(I,J) = R1M(1,J)	SL039300
RST(I,IK) = S1M(1,J)	SL039400
RST(I,IL) = T1M(1,J)	SL039500
RST(JK,J) = R2M(1,J)	SL039600
RST(JK,IK) = S2M(1,J)	SL039700
RST(JK,IL) = T2M(1,J)	SL039800
RST(JL,J) = R3M(1,J)	SL039900
RST(JL,IK) = S3M(1,J)	SL040000
RST(JL,IL) = T3M(1,J)	SL040100
190 CONTINUE	SL040200
IJ = 3*N+1	SL040300
IK = 3*N+2	SL040400
IL = 3*N+3	SL040500
UVW(IJ,IJ) = X4M	SL040600
UVW(IJ,IK) = Y4M	SL040700
UVW(IJ,IL) = Z4M	SL040800
UVW(IK,IJ) = X5M	SL040900
UVW(IK,IK) = Y5M	SL041000
UVW(IK,IL) = Z5M	SL041100
UVW(IL,IJ) = X6M	SL041200
UVW(IL,IK) = Y6M	SL041300
UVW(IL,IL) = Z6M	SL041400
RST(IJ,IJ) = Q4M	SL041500
RST(IJ,JK) = P4M	SL041600
RST(IJ,IL) = Q4M	SL041700
RST(IK,IJ) = Q5M	SL041800
RST(IK,IK) = P5M	SL041900
RST(IK,IL) = Q5M	SL042000
RST(IL,IJ) = Q6M	SL042100
RST(IL,IK) = P6M	SL042200
RST(IL,IL) = Q6M	SL042300
IF (NOPT)4000,4005,4000	SL042400
4000 PRINT 4010	SL042500
4010 FORMAT (1H1,*, (U) MATRIX *)	SL042600
CALL MPRINT (UVW,N3,N3,NDIM)	SL042700
PRINT 4015	SL042800
4015 FORMAT (1H1,*, (R) MATRIX *)	SL042900
CALL MPRINT (RST,N3,N3,NDIM)	SL043000
4005 CONTINUE	SL043100
IF (M)198,199,198	SL043200
198 CONTINUE	SL043300
CALL MATINV (UVW,IROW,ICOL,N3,NDIM,SMLST)	SL043400
DO 192 I=1,N3	SL043500
DO 192 J=1,N3	SL043600
SUM = 0,	SL043700
DO 191 K=1,N3	SL043800

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      SUM      = SUM      + UVW(I,K)*RST(K,J)
191 CONTINUE
      UTR(I,J) = SUM
      UTRSV(I,J) = UTR(I,J)
192 CONTINUE
      IF (NOPT)4020,4025,4020
4020 PRINT 4030
4030 FORMAT (1H1,* ( U ) MATRIX INVERSE *)
      CALL MPRINT (UVW,N3,N3,NDIM)
      PRINT 4035
4035 FORMAT (1H1,* (U) INVERSE X (R) *)
      CALL MPRINT (UTR,N3,N3,NDIM)
4025 CONTINUE
      CALL MITERS (UTR,NTAPE,N3,GUESS,NGUESS,NMODE,VECTOR,EIGVAL,
1 NITER,NITRSP,EPSP,US,H4,MAXR,NC,AITKEN,NAKSR,UTRSV)
      DO 194 I=1,1
      XLDA = EIGVAL(I)
      TM1 = (CS/A)/SQRT(1.-ENU**2)
      OMEGR = SQRT(ABS(1./XLDA))*TM1
      OMEGC = OMEGR/(2.*PI)
      IF ( ITER )618,619,619
619 EIGCPS(I) = OMEGC
618 PRINT 615, I,XLDA,OMEGR,OMEGC
615 FORMAT (15,3E20,8)
194 CONTINUE
      W = EIGCPS(ITER )
      IF ( ITER )197,7000,197
197 WCPS(ITER) = (SQRT(ABS(1./EIGVAL(ITER)))*TM1)/(2.*PI)
      DO 196 I=1,N3
      AMNK(I,ITER) = VECTOR(I,ITER)
196 CONTINUE
7000 CONTINUE
      KOPT = 1
      GO TO 3
199 CONTINUE
      DO 195 I=1,N3
      DO 195 J=1,N3
      UWR(I,J) = UVW(I,J)-WKBSQ*RST(I,J)
195 CONTINUE
      DO 600 I=1,N
      J = N+I
      K = 2*N+I
      AI(I) = ((-CSCOW*SINF(ALNP(I)*H)+ALNP(I)*SINF(CSCOW*H))/
1 (ALNP(I)**2-CSCOW**2))/AONSO
      QBH(I) = (RHO*A*OMEGA*C0*AI(I))/(RHOS*SHS*CS*COSF(CSCOW*H))
      FH(I) = (1.-ENU**2)*QBH(I)
      FH(J) = 0.
      FH(K) = 0.
600 CONTINUE
      FH(3*N+1) = +1.
      FH(3*N+2) = 0.
      FH(3*N+3) = 0.
      IF (NOPT)4045,4050,4045
4045 PRINT 4055
4055 FORMAT (1H1,* (U) - OMEGABSQ (R) *)
      CALL MPRINT (UWR,N3,N3,NDIM)
      PRINT 4060
4060 FORMAT (*1 MATRIX (I)*)
      PRINT 4065,(AI(I),I=1,N)
4065 FORMAT (5E20,8)
      PRINT 4070
4070 FORMAT (*0 MATRIX OHATB(I)*)

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      PRINT 4065, (GRH(I), I=1, N)
      PRINT 4075
4075  FORMAT (*0 MATRIX (F)*)
      PRINT 4065, (FH(I), I=1, N3)
4050  CONTINUE
      CALL MATINV (UWR, IROW, ICOL, N3, NDIM, SMLST)
      DO 605 I=1, N3
      APH(I) = 0.
      DO 610 J=1, N3
      APH(I) = APH(I) + UWR(I, J) * FH(J)
610  CONTINUE
605  CONTINUE
      IF (NOPT) 4080, 4085, 4080
4080  PRINT 4090
4090  FORMAT (*0 APHAT(I) *)
      PRINT 4065, (APH(I), I=1, N3)
      PRINT 4065, (ALN(I), I=1, N)
      PRINT 4065, (X1(I), I=1, N)
      PRINT 4065, (X2(I), I=1, N)
      PRINT 4065, (X0B(I), I=1, N)
      PRINT 4065, (X1R(I), I=1, N)
      PRINT 5040
5040  FORMAT (1H1, *(U) = OMEGABSQ (R), INVERSE *)
      CALL MPRINT (UWR, N3, N3, NDIM)
4085  CONTINUE
      R0PH = APH(3*N+1)
      R1PH = APH(3*N+2)
      R2PH = APH(3*N+3)
      DO 225 I=1, N
      J = N+I
      CN1(I) = B2PH*X1R(I) + B1PH*X0B(I) - ALN(I)*APH(J) + ENU*APH(I)
      CN2(I) = APH(I) + ENU*(B2PH*X1R(I) + B1PH*X0B(I) - ALN(I)*APH(J))
      CN3(I) = ENU*ALN(I)*APH(I) - ALN(I)**2*APH(J)
225  CONTINUE
      DO 230 I=1, N
      DO 230 J=1, N
      DO 230 K=1, N
      IF ( I-J+K ) 805, 800, 805
800  TERM1 = 0.
      TERM5 = 0.
      TERM2 = (-1+(-1)**(J+K-I))/(ALN(J)+ALN(K)-ALN(I))
      TERM3 = (-1+(-1)**(I+J-K))/(ALN(I)+ALN(J)-ALN(K))
      GO TO 830
805  IF ( J+K-I ) 815, 810, 815
810  TERM2 = 0.
      TERM1 = (-1+(-1)**(I-J+K))/(ALN(I)-ALN(J)+ALN(K))
      TERM3 = (-1+(-1)**(I+J-K))/(ALN(I)+ALN(J)-ALN(K))
      TERM5 = (1.-(-1)**(J-I-K))/(ALN(J)-ALN(I)-ALN(K))
      GO TO 830
815  IF ( I+J-K ) 825, 820, 825
820  TERM3 = 0.
      TERM1 = (-1+(-1)**(J-J+K))/(ALN(I)-ALN(J)+ALN(K))
      TERM2 = (-1+(-1)**(J+K-I))/(ALN(J)+ALN(K)-ALN(I))
      TERM5 = (1.-(-1)**(J-I-K))/(ALN(J)-ALN(I)-ALN(K))
      GO TO 830
825  TERM1 = (-1+(-1)**(I-J+K))/(ALN(I)-ALN(J)+ALN(K))
      TERM2 = (-1+(-1)**(J+K-I))/(ALN(J)+ALN(K)-ALN(I))
      TERM3 = (-1+(-1)**(I+J-K))/(ALN(I)+ALN(J)-ALN(K))
      TERM5 = (1.-(-1)**(J-I-K))/(ALN(J)-ALN(I)-ALN(K))
830  TERM4 = (+1-(-1)**(I+J+K))/(ALN(I)+ALN(J)+ALN(K))
      DNNN(I, J, K) = -( .25/AONSQ)*(TERM1+TERM2+TERM3+TERM4)
      ENNN(I, J, K) = ( .25/AONSQ)*(TERM4+TERM5-TERM3-TERM2)

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SL050100
 SL050200
 SL050300
 SL050400
 SL050500
 SL050600
 SL050700
 SL050800
 SL050900
 SL051000
 SL051100
 SL051200
 SL051300
 SL051400
 SL051500
 SL051600
 SL051700
 SL051800
 SL051900
 SL052000
 SL052100
 SL052200
 SL052300
 SL052400
 SL052500
 SL052600
 SL052700
 SL052800
 SL052900
 SL053000
 SL053100
 SL053200
 SL053300
 SL053400
 SL053500
 SL053600
 SL053700
 SL053800
 SL053900
 SL054000
 SL054100
 SL054200
 SL054300
 SL054400
 SL054500
 SL054600
 SL054700
 SL054800
 SL054900
 SL055000
 SL055100
 SL055200
 SL055300
 SL055400
 SL055500
 SL055600
 SL055700
 SL055800
 SL055900
 SL056000
 SL056100
 SL056200

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230 CONTINUE
DO 310 J=1,N
DO 310 K=1,N
TERM1 = (1.-(-1)**(J*K))/(ALN(J)*ALN(K))
IF (J-K) 840,835,840
835 TERM2 = 0.
GO TO 845
840 TERM2 = (1.-(-1)**(J-K))/(ALN(J)*ALN(K))
845 E0NN(J,K) = (.25/AON SQ) * (2.*(TERM1+TERM2))
310 CONTINUE
M = 10
DO 235 I=1,N
DO 235 J=1,N
SUM = 0.
DO 240 K=1,N
SUM = SUM + (ALN(J)**2*CN1(K)*DNNN(K,I,J)-ALN(J)*CN3(K)*ENNN(K,I,J)
1 +M*M*CN2(K)*DNNN(K,I,J))
240 CONTINUE
TPNN(I,J) = (SUM-ALN(J)*B2PH*E0NN(I,J))*AMNK(J,JTER)
235 CONTINUE
IF (NOPT) 4095,5000,4095
4095 PRINT 5005
5005 FORMAT (1H1,12X,*N1(I)*,15X,*N2(I)*,15X,*N3(I)*)
PRINT 5010, (I,CN1(I),CN2(I),CN3(I),I=1,N)
5010 FORMAT (15,3E20,8)
PRINT 5035
5035 FORMAT (*1 TP(I,J) *)
CALL MPRINT (TPNN,N,N,N)
5000 SUM1 = SUM2 = 0.
DO 245 I=1,N
SUM3 = SUM4 = 0.
DO 250 J=1,N
SUM3 = SUM3+AMNK(J,JTER)*BMMNN0(JTER,I,J)
SUM4 = SUM4+AMNK(J,JTER)*BMMNNK(JTER,I,J)
250 CONTINUE
SUM1 = SUM1+AMNK(I,JTER)**2+AMNK(I,JTER)*SUM3
SUM2 = SUM2+AMNK(I,JTER)**2+AMNK(I,JTER)*SUM4
245 CONTINUE
RMBAR = WKBSQ*SUM1
BKBAR = WKSQ*SUM2
BTBAR = 0.
DO 255 I=1,N
DO 255 J=1,N
BTRAR = BTBAR+TPNN(I,J)*AMNK(I,JTER)
255 CONTINUE
ABAR = (4.*BKBAR)/RMBAR
QBAR = (2.*BTBAR)/RMBAR
X0AQ = ARSF((1.-ABAR)/QBAR)
X0STR = X0AQ*((WRAD**2*A)/G)
WSTR = W/(2.*WCPS(JTER))
PRINT 4040, W,WKB,BKBAR,BTRAR,RMBAR,ABAR,QBAR,X0AQ,X0STR,WSTR
4040 FORMAT (1H0,2F10.2,8E12.3)
GO TO 1000
END
FUNCTION DKN(KD,ND)
COMMON PI,H/COM4/ALN(10)
IF (KD)10,15,10
15 DKN = (1.0-COSF(ALN(ND)*H))/(H*ALN(ND))
GO TO 20
10 DKN = ((2.0*ALN(ND))/(H*(ALN(ND)**2-((KD*PI)/H)**2)))*(1.0-
1 (-1)**KD*COSF(ALN(ND)*H))
20 RETURN

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END
FUNCTION DKNB(KDB,NPD)
COMMON PI,H/COM3/AONSQ/COM5/ALNP(10)
IF (KDB)10,15,10
15 DKNB = (1.0-COSF(ALNP(NPD)*H))/(AONSQ*ALNP(NPD))
GO TO 20
10 DKNB = (ALNP(NPD)/AONSQ)*((1.0-(-1)**KDB*CCSF(ALNP(NPD)*H))/
1 (ALNP(NPD)**2-((KDB*PI)/H)**2))
20 RETURN
END
FUNCTION RMK (MR,KR)
COMMON PI,H/COM1/A,A0,CSC0W
DIMENSION T(1000)
XIK = SQRTF(ABSF(((KR*PI*A0)/(H*A))**2-CSC0W**2))
IF (KR-(CSC0W*H)/PI)10,10,15
10 CALL BES (MR,XIK,0,RJM,T)
CALL BES (MR+1,XIK,0,RJM1,T)
RMK = RJM/(+MR*RJM-XIK*RJM1)
GO TO 20
15 CALL BES (MR,XIK,1,RIM,T)
CALL BES (MR+1,XIK,1,RIM1,T)
RMK = RIM/(+MR*RIM+XIK*RIM1)
20 RETURN
END
FUNCTION EMKJ (ME,KE,JE)
COMMON PI,H/COM1/A,A0,CSC0W/COM2/UMJ( 5)
DIMENSION T(1000)
CALL BES (ME,UMJ(JE),0,EJM,T)
ALFSQ = 0.5*(1.0-(ME**2/UMJ(JE)**2))*EJM**2
XIK = SQRTF(ABSF(((KE*PI*A0)/(H*A))**2-CSC0W**2))
IF (KE-(CSC0W*H)/PI)10,15,15
10 EMKJ = +EJM/((XIK**2-UMJ(JE)**2)*ALFSQ)
GO TO 20
15 EMKJ = +EJM/((XIK**2+UMJ(JE)**2)*ALFSQ)
20 RETURN
END
SUBROUTINE BES(NO,X,KODE,RESULT,T)
03 UCSD RES      BESSEL FUNCTION
C      C3 UCSD RES
      DIMENSION T(1000)
107  FORMAT(55H1NEGATIVE ORDER NOT ACCEPTED IN BESSEL FUNCTION ROUTINE)
      KLAM=1
      KO=NO+1
      IF(X) 6,1,6
1      IF(NO) 4,2,3
2      T(KO)=1.0
      RESULT=1.0
      RETURN
4      IF(KO) 5,10,3
3      RESULT=0
      RETURN
10     RESULT=9.999999999E200
      RETURN
5      PRINT 107
      STOP1
6      IF(NO) 5,7,7
7      IF(KODE) 8,9,8
8      KLAM=KLAM+1
9      JO=2*IFIX(X)
      MO=NO
      IF(MO-JO) 11,12,12
11     MO=JO

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DK000900
DK000100
DK000200
DK000300
DK000400
DK000500
DK000600
DK000700
DK000800
DK000900
RM000100
RM000200
RM000300
RM000400
RM000500
RM000600
RM000700
RM000800
RM000900
RM001000
RM001100
RM001200
RM001300
RM001400
EM000100
EM000200
EM000300
EM000400
EM000500
EM000600
EM000700
EM000800
EM000900
EM001000
EM001100
EM001200
BES      1
F-63
F 63
BES      2
BES      3
BES      4
BES      5
BES      6
BES      7
BES      8
BES      9
BES     10
BES     11
BES     12
BES     13
BES     14
BES     15
BES     16
BES     17
BES     18
BES     19
BES     20
BES     21
BES     22
BES     23
BES     24

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12	M0=M0+11	BES	25
	T(M0)=0,	BES	26
	LUB=M0-1	BES	27
	T(LUB)=1.0E+250	BES	28
	GO TO (23,51),KLAM	BES	29
23	F=2*LUB	BES	30
	M0=M0-3	BES	31
	I2=M0	BES	32
24	F=F-2,	BES	33
	T(I2+1)=F/X*T(I2+2)+T(I2+3)	BES	34
	IF(I2)25,26,25	BES	35
25	I2=I2-1	BES	36
	GO TO 24	BES	37
26	SUM=T(1)	BES	38
	DO 40 J=3,M0,2	BES	39
40	SUM=SUM+2.*T(J)	BES	40
	F=1./SUM	BES	41
	DO 50 J= 1,K0	BES	42
50	T(J)=T(J)*F	BES	43
	RESULT=T(K0)	BES	44
	RETURN	BES	45
51	F=2*LUB+2	BES	46
	M0=M0-3	BES	47
	I2=M0	BES	48
511	T(I2+1)=F/X*T(I2+2)+T(I2+3)	BES	49
	IF(I2)52,53,52	BES	50
52	I2=I2-1	BES	51
	F=F-2.	BES	52
	GO TO 511	BES	53
53	SUM=T(1)	BES	54
	DO 70 J=2,M0	BES	55
70	SUM=SUM+2.*T(J)	BES	56
	F=1./SUM*EXP(X)	BES	57
	DO 80 J=1,K0	BES	58
80	T(J)=T(J)*F	BES	59
	RESULT=T(K0)	BES	60
	RETURN	BES	61
	END	BES	62
	SUBROUTINE MATINV (A , IROW , ICOL , N , NDIM , SMLST)	MA	100
	DIMENSION A (1) , IROW (1) , ICOL (1)	MA	200
C	709-16065	MA	300
C	709-16065 SUBROUTINE MATINV - MATRIX INVERSION ROUTINE	MA	400
C		MA	500
C	A = ARRAY NAME OF MATRIX	MA	600
C	IROW = DIMENSIONED AT N+1 OR GREATER	MA	700
C	ICOL = DIMENSIONED AT N OR GREATER	MA	800
C	N = NUMBER OF EQUATIONS	MA	900
C	NDIM = VALUE OF I IN DIMENSION A(I,J) , I AND J MAY DIFFER	MA	1000
C	SMLST = SMALLEST LEADING ELEMENT ALLOWED BEFORE CALLING THE	MA	1100
C	SYSTEM SINGULAR , USUALLY = 1.0 E-04 OR 1.0 E-05	MA	1200
C		MA	1300
	NP1 = N + 1	MA	1400
	DO 100 I = 1, N	MA	1500
	ICOL (I) = I	MA	1600
100	IROW (I) = I	MA	1700
	DO 240 ITER = 1, N	MA	1800
	MAXR = ITER	MA	1900
	MAXC = 1	MA	2000
	TEMP = ABSF (A (MAXR))	MA	2100
	LIMITC = NP1 - ITER	MA	2200
	DO 120 I = ITER, N	MA	2300
	DO 120 J = 1, LIMITC	MA	2400

	IJ = (J - 1) * NDIM + I	MA 2500
	IF (TEMP - (ARSF (A (IJ)))) 110, 120, 120	MA 2600
C		MA 2700
110	MAXR = I	MA 2800
	MAXC = J	MA 2900
	TEMP = ARSF (A (IJ))	MA 3000
120	CONTINUE	MA 3100
	IF (TEMP - SMLST) 130, 130, 140	MA 3200
C		MA 3300
130	IROW (NP1) = ITER	MA 3400
	PRINT 1, ITER , SMLST	MA 3500
	RETURN	MA 3600
C		MA 3700
140	IF (MAXR - ITER) 150, 170, 150	MA 3800
C		MA 3900
150	DO 160 J = 1, N	MA 4000
	MAXRJ = (J - 1) * NDIM + MAXR	MA 4100
	ITJ = (J - 1) * NDIM + ITER	MA 4200
	TEMP = A (MAXRJ)	MA 4300
	A (MAXRJ) = A (ITJ)	MA 4400
160	A (ITJ) = TEMP	MA 4500
	ITEMP = IROW (MAXR)	MA 4600
	IROW (MAXR) = IROW (ITER)	MA 4700
	IROW (ITER) = ITEMP	MA 4800
170	IF (MAXC - 1) 180, 200, 180	MA 4900
C		MA 5000
180	DO 190 I = 1, N	MA 5100
	IMAXC = (MAXC - 1) * NDIM + I	MA 5200
	TEMP = A (I)	MA 5300
	A (I) = A (IMAXC)	MA 5400
190	A (IMAXC) = TEMP	MA 5500
	ITEMP = ICOL (MAXC)	MA 5600
	ICOL (MAXC) = ICOL (1)	MA 5700
	ICOL (1) = ITEMP	MA 5800
200	TEMP = A (ITER)	MA 5900
	ITEMP = ICOL (1)	MA 6000
	DO 210 J = 2, N	MA 6100
	ITJM1 = (J - 2) * NDIM + ITER	MA 6200
	ITJ = (J - 1) * NDIM + ITER	MA 6300
	A (ITJM1) = A (ITJ) / TEMP	MA 6400
210	ICOL (J - 1) = ICOL (J)	MA 6500
	ITN = (N - 1) * NDIM + ITER	MA 6600
	A (ITN) = 1 . 0 / TEMP	MA 6700
	ICOL (N) = ITEMP	MA 6800
	DO 240 I = 1, N	MA 6900
	IF (I = ITER) 220, 240, 220	MA 7000
C		MA 7100
220	TEMP = A (I)	MA 7200
	DO 230 J = 2, N	MA 7300
	IJM1 = (J - 2) * NDIM + I	MA 7400
	IJ = (J - 1) * NDIM + I	MA 7500
	ITJM1 = (J - 2) * NDIM + ITER	MA 7600
	A (IJM1) = A (IJ) - A (ITJM1) * TEMP	MA 7700
230	CONTINUE	MA 7800
	IN = (N - 1) * NDIM + I	MA 7900
	ITN = (N - 1) * NDIM + ITER	MA 8000
	A (IN) = - (TEMP * A (ITN))	MA 8100
240	CONTINUE	MA 8200
	DO 290 I = 1, N	MA 8300
	DO 250 J = I, N	MA 8400
	IF (IROW (J) - I) 250, 260, 250	MA 8500
C		MA 8600

250	CONTINUE	MA	8700
260	IF (I = J) 270, 290, 270	MA	8800
C		MA	8900
270	DO 280 L = 1, N	MA	9000
	LI = (I - 1) * NDIM + L	MA	9100
	LJ = (J - 1) * NDIM + L	MA	9200
	TEMP = A (LI)	MA	9300
	A (LI) = A (LJ)	MA	9400
280	A (LJ) = TEMP	MA	9500
	IROW (J) = IROW (I)	MA	9600
290	CONTINUE	MA	9700
	DO 340 I = 1, N	MA	9800
	DO 300 J = I, N	MA	9900
	IF (ICOL (J) = I) 300, 310, 300	MA	10000
C		MA	10100
300	CONTINUE	MA	10200
310	IF (I = J) 320, 340, 320	MA	10300
C		MA	10400
320	DO 330 L = 1, N	MA	10500
	IL = (L - 1) * NDIM + I	MA	10600
	JL = (L - 1) * NDIM + J	MA	10700
	TEMP = A (IL)	MA	10800
	A (IL) = A (JL)	MA	10900
330	A (JL) = TEMP	MA	11000
	ICOL (J) = ICOL (I)	MA	11100
340	CONTINUE	MA	11200
	IROW (NP1) = N	MA	11300
	RETURN	MA	11400
C		MA	11500
	1 FORMAT (7H00N THE13,63HTH ITERATION ALL THE REMAINING TERMS WERE LMA	MA	11600
	*ESS THAN OR EQUAL TO E11.4,18H IN ABSOLUTE VALUE)	MA	11700
	END	MA	11800
	SUBROUTINE MPRINT (A , M , N , MD)	MP	100
C	MATRIX PRINT SUBROUTINE	MP	200
C	THE CALL FOR THIS SUBROUTINE IS AS FOLLOWS,	MP	300
C	CALL MPRINT (A,M,N,MD)	MP	400
C	WHERE A IS THE MATRIX TO BE PRINTED	MP	500
C	M IS THE NUMBER OF ROWS	MP	600
C	N IS THE NUMBER OF COLUMNS	MP	700
C	MD IS DIMENSIONED NO. OF ROWS OF MATRIX A	MP	800
	DIMENSION A (1) , JT (6) , C (6)	MP	900
	EQUIVALENCE (JT , C)	MP	1000
	N1 = N	MP	1100
	N2 = 6	MP	1200
	N3 = 6	MP	1300
	N4 = 1	MP	1400
100	IF (N3 = N1) 120, 120, 110	MP	1500
C		MP	1600
110	N2 = N1 - N3 + 6	MP	1700
	N3 = N1	MP	1800
120	K = 0	MP	1900
	DO 130 I = N4, N3	MP	2000
	K = K + 1	MP	2100
130	JT (K) = I	MP	2200
	PRINT 1, (JT (I) , I = 1 , N2)	MP	2300
	DO 150 I = 1, N	MP	2400
	K = 0	MP	2500
	L = MD + (N4 - 1) + I	MP	2600
	DO 140 J = N4, N3	MP	2700
	K = K + 1	MP	2800
	C (K) = A (L)	MP	2900
140	L = L + MD	MP	3000

150 PRINT 2, I, (C (K) , K = 1 , N2)	MP 3100
IF (N3 - N1) 160, 170, 170	MP 3200
C	MP 3300
160 N3 = N3 + 6	MP 3400
N4 = N4 + 6	MP 3500
GO TO 100	MP 3600
C	MP 3700
170 RETURN	MP 3800
C	MP 3900
1 FORMAT (1H , 4X, 6(6X, 7HCOLUMN 114) /)	MP 4000
2 FORMAT (1H 114, X, (6E 17,8))	MP 4100
END	MP 4200
SUBROUTINE NPNRMX (A, B, N, FL, INDEX, MD, NX)	MTRS0148
CALL NPNRMX, (A, B, N, FL, INDEX, MD, NX, NP)	MTRS0135
A=VECTOR TO BE NORMALIZED B=NORMALIZED VECTOR(MAY=A)	MTRS0136
N=SIZE FL=NORMALIZING NUMBER	MTRS0137
INDEX=+ ON ENTRY, NORMALIZE ON NUMBER WHOSE INDEX IS INDEX	MTRS0138
=0 ON ENTRY, NORMALIZE ON LARGEST S.W. AND SET	MTRS0139
INDEX=TO ITS INDEX,	MTRS0140
=- ON ENTRY, NORMALIZE ON FL.	MTRS0141
MD=SINGLE PRECISION DIMENSIONED NUMBER OF ROWS OF A AND B	MTRS0142
NX=1, VECTOR REAL	MTRS0143
=2, VECTOR COMPLEX	MTRS0144
NP=1, SINGLE PRECISION	MTRS0145
C	MTRS0150
DIMENSION A(1), B(1), FL(1), D(1), C(1)	MTRS0152
N1 = 2	MTRS0153
N2=N	MTRS0154
N4=MD	MTRS0155
IF (INDEX) 32, 7, 38	MTRS0156
7 GOTO (11,8),NX	MTRS0157
8 FL= (A(1)**2+A(N4+1)**2)	MTRS0158
INDEX=1	MTRS0259
DO 10 K=N1,N2	MTRS0160
I=K+N4	MTRS0161
D= (A(K)**2+A(I)**2)	MTRS0162
IF (FL-D) 9,9,10	MTRS0163
9 FL =D	MTRS0164
INDEX=K	MTRS0165
10 CONTINUE	MTRS0166
6 FL=A(INDEX)	MTRS0167
GOTO 18	MTRS0169
11 FL=ABSF(A(1))	MTRS0170
INDEX=1	MTRS0171
DO 13 K=N1,N2	MTRS0172
D=ABSF(A(K))	MTRS0173
IF (FL-D) 12,12,13	MTRS0174
12 FL=D	MTRS0175
INDEX=K	MTRS0176
13 CONTINUE	MTRS0178
14 FL=A(INDEX)	MTRS0180
16 DO 17 I=1,N	MTRS0181
17 B(I)=A(I)/FL	MTRS0182
GOTO 30	MTRS0184
18 I=INDEX+MD	MTRS0185
FL(2)=A(I)	MTRS0186
19 D=FL(1)**2+FL(2)**2	MTRS0187
DO 20 J=1,N	MTRS0188
K=J+MD	MTRS0189
C=A(I)*FL(2)-A(K)*FL(1)	MTRS0190
B(I)=(A(I)*FL(1)+A(K)*FL(2))/D	MTRS0191
20 B(K)=-C/D	MTRS0211
30 RETURN	

32 GOTO (34,36), NX	MTRS0213
34 GOTO 16	MTRS0214
36 GOTO 19	MTRS0215
38 GOTO 40	MTRS0217
40 GOTO (14,6), NX	MTRS0219
END	MTRS0220
SUBROUTINE DPMLTX (A,NA,B,NB,C,M,N,K,MA,MB,MC)	MTRS0053
SUBROUTINE	MTRS0035
CALLING SEQUENCE.....	MTRS0037
CALL DPMLTX (A,NA,B,NB,C,M,N,K,MA,MB,MC)	MTRS0038
A = PREMULTIPLIER MA = DIMENSIONED NUMBER OF ROWS A	MTRS0040
B = POSTMULTIPLIER MB = DIMENSIONED NUMBER OF ROWS B	MTRS0041
C = PRODUCT MC = DIMENSIONED NUMBER OF ROWS C	MTRS0042
M = NO. ROWS IN A = NO. ROWS IN C	MTRS0043
N = NO. COLUMNS IN A = NO. ROWS IN B	MTRS0044
K = NO. COLUMNS IN B = NO. COLUMNS IN C	MTRS0045
NA AND NB = 1 IF A OR B, RESPECTIVELY, ARE REAL.	MTRS0046
= 2 IF A OR B, RESPECTIVELY, ARE COMPLEX.	MTRS0047
MA,MB,MC ARE SINGLE PRECISION DIMENSIONS,(1/2 OF ACTUAL CORE	MTRS0049
RESERVED FOR REAL DOUBLE PRECISION MATRICES, AND 1/4 OF	MTRS0050
ACTUAL CORE RESERVED FOR COMPLEX DOUBLE PRECISION MATRICES)	MTRS0051
DIMENSION A(1), B(1), C(1)	MTRS0055
IA=MC*K	MTRS0057
IB=MA*N+NA	MTRS0058
IC=1	MTRS0059
ID=MA*NA	MTRS0060
IH=MC	MTRS0061
IJ=MC	MTRS0062
IK=M	MTRS0063
IL=1	MTRS0064
IM=0	MTRS0065
GOTO (6,7),NA	MTRS0066
6 GOTO (11,10),NB	MTRS0067
7 GOTO (9,8),NB	MTRS0068
8 IC=2	MTRS0069
GOTO 10	MTRS0070
9 IH=2*IH	MTRS0071
IC=3	MTRS0072
10 IA=2*IA	MTRS0073
IJ=2*IJ	MTRS0074
11 DO 30 I=1,IK,IL	MTRS0075
INC=IM	MTRS0076
DO 16 J=I,IA,IH	MTRS0077
IN=INC	MTRS0078
12 C(J)=0.	MTRS0081
DO 13 L=I,IB,ID	MTRS0082
IN=IN+1	MTRS0083
13 C(J)=C(J)+A(L)*B(IN)	MTRS0084
16 INC=INC+MB	MTRS0091
GOTO (30,18,24),IC	MTRS0093
18 IE=I+MA	MTRS0094
INC=IM	MTRS0095
DO 23 J=I,IA,IJ	MTRS0096
IN=INC	MTRS0097
IF=J+MC	MTRS0098
19 DO 20 L=IE,IB,ID	MTRS0100
IN=IN+1	MTRS0101
IG=IN+MB	MTRS0102
C(IF)=C(IF)+A(L)*B(IN)	MTRS0103
20 C(J)=C(J)-A(L)*B(IG)	MTRS0104
23 INC=INC+2*MB	MTRS0112
GOTO 30	MTRS0113

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24   IE=I+MC
      IF=I+MA
      INC=IM
      DO 29 J=IE,JA,IJ
      IN=IAC
25   C(J)=0.
      DO 26 L=IF,IH,ID
      IN=IN+1
26   C(L)=C(J)+A(L)*B(IN)
29   INC=INC+MB
30   CONTINUE
      RETURN
      END
      SUBROUTINE SWEEPX (HTRUE, U, H, US, FL, MODE, N, MD, NC, INDEX, EP)
C   COMPUTES TRUE MODE AND SWEEPS IT FROM THE MATRIX. (REAL OR COMPLEX)
C   HTRUE = TRUE MODAL COLUMNS, AS COMPUTED. U = DYNAMIC MATRIX,
C   H = SERIES OF MODIFIED MODAL COLUMNS, FL = COLUMN OF EIGENVALUES.
C   US = SERIES OF MODIFIED MODAL ROWS OF U.
C   MODE = MODE NOW BEING COMPUTED. N = SIZE
C   MD = DIMENSIONED NUMBER OF ROWS OF U, US, H, HTRUE
C   NX = 1 IF PROBLEM IS REAL.
C   = 2 IF PROBLEM IS COMPLEX.
      DIMENSION H(1), US(1), U(1), HTRUE(1), FL(1), G(4)
      M=MODE-1
      K1=M*NC+MD
      DO 6 J=1,NC
      K=K1+(J-1)*MD
      DO 6 L=1,N
      K=K+1
6     HTRUE(K)=H(K)
      IF (M) 31,31,8
8     DO 25 J=1,M
      L1=NC*MD+(MODE-1) -NC*MD
      GOTO (9,11),NC
9     G=0.
      DO 10 J=1,N
      L=L1+J
      K=K1+J
10    G=G+US(L)*HTRUE(K)
      GOTO 13
11    G(1)=0.
      G(2)=0.
      DO 12 J1=1,N
      L=L1+J1
      K=K1+J1
      L2=L+MD
      K2=K+MD
      G(1)=G(1)+US(L)*HTRUE(K)-US(L2)*HTRUE(K2)
12    G(2)=G(2)+US(L)*HTRUE(K2)+US(L2)*HTRUE(K)
13    K=MODE-1
      GOTO (14,19),NC
14    IF (ABS(FL(K)/FL(MODE)-1.) - EP) 15,15,16
15    G=1.
      GOTO 17
16    G=(FL(K)-FL(MODE)) / G
17    DO 18 J=1,N
      K=K1+J
      L=L1+J
18    HTRUE(K)=H(L)-G(1)*HTRUE(K)
      GOTO 25
19    K=2*K
      J=2*MODE

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MTRS0114
MTRS0115
MTRS0116
MTRS0117
MTRS0118
MTRS0120
MTRS0121
MTRS0122
MTRS0123
MTRS0129
MTRS0131
MTRS0132
MTRS0133
MTRS0233
MTRS0223
MTRS0225
MTRS0226
MTRS0227
MTRS0228
MTRS0229
MTRS0230
MTRS0231
MTRS0235
MTRS0237
MTRS0238
MTRS0240
MTRS0241
MTRS0242
MTRS0243
MTRS0244
MTRS0246
MTRS0247
MTRS0248
MTRS0249
MTRS0250
MTRS0251
MTRS0252
MTRS0253
MTRS0254
MTRS0255
MTRS0256
MTRS0257
MTRS0258
MTRS0259
MTRS0260
MTRS0261
MTRS0262
MTRS0263
MTRS0264
MTRS0265
MTRS0266
MTRS0267
MTRS0268
MTRS0269
MTRS0270
MTRS0271
MTRS0272
MTRS0273
MTRS0274
MTRS0275
MTRS0276
MTRS0277

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      IF ( ABSF((FL(K-1)*FL(J-1)+FL(K)*FL(J))/(FL(J-1)**2+FL(J)**2)-1.) MTRS0278
1      *EP) 20,20,22 MTRS0279
20 IF ( ABSF((FL(K)*FL(J-1)-FL(K-1)*FL(J)) / (FL(J-1)**2+FL(J)**2)) MTRS028
1      -EP) 21,21,22 MTRS0281
21 G(1)=1. MTRS0282
   G(2)=0. MTRS0283
   GOTO 23 MTRS0284
22 G(3)=G(1)**2+G(2)**2 MTRS0285
   G(4)=(FL(K)-FL(J))*G(1)-(FL(K-1)-FL(J-1))*G(2) MTRS0286
   G(1)=((FL(K-1)-FL(J-1))*G(1)+(FL(K)-FL(J))*G(2)) / G(3) MTRS0287
   G(2)= G(4) / G(3) MTRS0288
23 DO 24 J1=1,N MTRS0290
   K=K1+J1 MTRS0291
   K2=K+ MD MTRS0292
   L=L1+J1 MTRS0293
   L2=L+MD MTRS0294
   G(3)=HTRUE(K) MTRS0295
   HTRUE(K)= H(L)+ G(2)*HTRUE(K2)-G(1)*HTRUE(K) MTRS0296
   HTRUE(K2)= H(L2)- G(1)*HTRUE(K2)-G(2)*G(3) MTRS0294
24 CONTINUE MTRS0298
25 CONTINUE MTRS0299
   I=0 MTRS0301
   CALL NPNRMX (HTRUE(K1+1),HTRUE(K1+1),N,C,I,MD,NC ) MTRS0302
31 GOTO (26,32),NC. MTRS0304
26 DO 29 J=1,N MTRS0305
   L1=(J-1)*MD MTRS0306
   L2=K1+J MTRS0307
   DO 29 I=1,N MTRS0308
   L=L1+I MTRS0309
   IF (I-INDEX) 28,27,28 MTRS0310
27 U(L)=0. MTRS0311
   GOTO 29 MTRS0312
28 K=K1+I MTRS0313
   U(L)=U(L)-H(K)*US(L2) MTRS0314
29 CONTINUE MTRS0315
30 RETURN MTRS0317
32 DO 35 I=1,N MTRS0319
   L1=MD+NC*(I-1) MTRS0320
   L2=K1+I MTRS0321
   J=L2+MD MTRS0322
   DO 35 J1=1,N MTRS0323
   L=L1+J1 MTRS0324
   K3=L+MD MTRS0325
   IF (J1-INDEX) 34,33,34 MTRS0326
33 U(L)=0. MTRS0327
   U(K3)=0. MTRS0328
   GOTO 35 MTRS0329
34 K=K1+J1 MTRS0330
   K2=K+MD MTRS0331
   U(I)=U(L)-H(K)*US(L2)+H(K2)*US(J) MTRS0332
   U(K3)=U(K3)-H(K2)*US(L2)-H(K)*US(J) MTRS0333
35 CONTINUE MTRS0334
   GOTO 30 MTRS0336
   END MTRS0337
SUBROUTINE MITERS (A, NTAPE, N, GUESS, NGUESS, NMODE, VECTOR, MTRS0375
1 EIGVAL, NITER, NITRSP, EPSP, MTRS0376
2 US, H,MAXR, NC, AITKEN, NAKSR,UTRSV) MTRS0377
C CALLING SEQUENCE..... MTRS0341
C A = MATRIX, DIMENSIONED (MAXR X 2*N) - REAL MTRS0343
C (MAXR X 2*N) - COMPLEX MTRS0344
C N = ORDER OF MATRIX MTRS0346
C GUESS=1ST, GUESS VECTOR, DIMENSIONED (MAXR X 1) - REAL MTRS0347

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C                                     (MAXR X 2) - COMPLEX      MTRS0348
C NGUESS=0, ROUTINE SUPPLIES GUESS VECTOR      MTRS0379
C      =+1, GUESS CONTAINS GUESS VECTOR      MTRS0350
C NMODE=NUMBER OF EIGEN SOLUTIONS REQUESTED    MTRS0351
C VECTOR=EIGENVECTORS, DIMENSIONED (MAXR X NMODE) - REAL  MTRS0352
C                                     (MAXR X 2*NMODE) - COMPLEX  MTRS0353
C EIGVAL=EIGENVALUES                          (NMODE X 1) - REAL  MTRS0354
C                                     (NMODE*2 X 1) - COMPLEX  MTRS0355
C NITER=NUMBER OF ITERATIONS PER MODE          MTRS0356
C NITRSP = MAXIMUM NUMBER OF SINGLE PREC. ITERATIONS      MTRS0357
C EPSP = CONVERGENCE CRITERIA FOR SINGLE ROOTS          MTRS0358
C US=CHECK EIGENVECTORS, DIMENSIONED (MAXR X NMODE) - REAL  MTRS0360
C                                     (MAXR X 2*NMODE) - COMPLEX  MTRS0361
C H=WORKING AREA OF CORE DIMENSIONED (MAXR X (NMODE+4) - REAL  MTRS0362
C                                     (MAXR X 2*(NMODE+4) - COMPLEX  MTRS0363
C                                     WILL CONTAIN CHECK EIGENVALUES, IF REQUESTED  MTRS0364
C MAXR = DIMENSIONED NUMBER OF ROWS      MTRS0367
C NC = 1, PROBLEM REAL      MTRS0368
C      = 2, PROBLEM COMPLEX      MTRS0369
C AITKEN = AITKEN CONVERGENCE CRITERIA    MTRS0370
C NAKSR = NUMBER OF TIMES AITKEN APPLIED    MTRS0371
C DIMENSION A(1), GUESS(1), VECTOR(1), EIGVAL(1), NITER(1), US(1),  MTRS0380
C      1      H(1), NAKSR(1), UTRSV(1)      MTRS0381
C DEFINE PROGRAM CONSTANTS AND ZEROS.      MTRS0408
C 8 MODE=0      MTRS0409
C   AT=AITKEN**2      MTRS0411
C   IF ( EPSP )      12,9,12      MTRS0412
C 9 FPSP = .1E-08      MTRS0413
C 12 IF ( NGUESS )      15,13,15      MTRS0417
C 13   J1=MAXR*(NC-1)      MTRS0419
C     DO 14 I=1,N      MTRS0420
C       K=J1+I      MTRS0421
C       GUESS(K)=0.      MTRS0422
C 14   GUESS(I)=1.      MTRS0423
C 15 MODE = MODE+1      MTRS0425
C     NAKSR(MODE)=0      MTRS0426
C     IGO=1      MTRS0428
C     NITER(MODE)=0      MTRS0429
C     K1=NC*MAXR*(MODE-1)      MTRS0430
C     K2=K1+1      MTRS0431
C     K3=NC*(MODE-1)+1      MTRS0432
C     K4= NC*MAXR      MTRS0433
C     K5= K4+NMODE      MTRS0434
C     K6=K5+K4      MTRS0435
C MOVE FIRST GUESS INTO POSITION      MTRS0437
C DO 16 J=1,NC      MTRS0438
C   J1=MAXR*(J-1)      MTRS0439
C   DO 16 I=1,N      MTRS0440
C     K=K1+J1+I      MTRS0441
C     L=J1+I      MTRS0442
C 16   H(K)=GUESS(L)      MTRS0443
C 17 NAK=0      MTRS0445
C 18 NITER(MODE)=NITER(MODE)+1      MTRS0446
C     NAK=NAK+1      MTRS0447
C     INDEX=0      MTRS0448
C     CALL DPLMTX (A,NC, H(K2),NC, VECTOR(K2), N,N,1, MAXR,MAXR,MAXR ) MTRS0449
C     CALL NPNRMX (VECTOR(K2), H(K2), N, EIGVAL(K3), INDEX, MAXR, NC ) MTRS0450
C TEST FOR SINGLE ROOT CONVERGENCE      MTRS0452
C DO 23 J=1,NC      MTRS0453
C   J1=(J-1)*MAXR      MTRS0454
C   K=K1+J1      MTRS0455
C   GOTO (24,19,21),NAK      MTRS0456

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19	L=K5+J1		MTRS0457
	DO 20 I=1,N		MTRS0458
	L=L+1		MTRS0459
	K=K+1		MTRS0460
	IF (ABSF(H(L)-H(K)) - EPSP)	20,20,24	MTRS0461
20	CONTINUE		MTRS0462
	GOTO 100		MTRS0463
21	DO 22 I=1,N		MTRS0464
	K=K+1		MTRS0465
	IF (ABSF(US(K)-H(K)) - EPSP)	22,22,24	MTRS0466
22	CONTINUE		MTRS0467
23	CONTINUE		MTRS0466
100	IF ACCUMULATOR OVERFLOW	108,102	MTRS0469
102	GOTO 56		MTRS0470
C	NO CONVERGENCE, SO TEST MAXIMUM NUMBER OF ITERATIONS.		MTRS0474
24	IF (NITER(MODE)-NITRSP)	25,46,46	MTRS0475
C	NOT YET EXCEEDED, SO TRY FOR AITKENS TIME.		MTRS0477
25	GOTO (40,44,31),NAK		MTRS0478
C	TEST FOR AITKENS CONVERGENCE.		MTRS0481
31	GOTO (26,36),NC		MTRS0482
26	DO 28 I=1,N		MTRS0484
	J=K5+I		MTRS0485
	K=K1+I		MTRS0486
	IF (US(K)-H(J))	27,261,27	MTRS0487
261	IF (H(K)-US(K))	32,28,32	MTRS0488
27	IF ((ABSF((H(K)-US(K))/ (US(K)-H(J)))) -AITKEN)	28,28,32	MTRS0489
28	CONTINUE		MTRS0490
C	ALL VECTOR ELEMENTS OK, SO APPLY AITKENS SPEEDER-UPPER.		MTRS0492
	DO 30 I=1,N		MTRS0493
	J=K5+I		MTRS0494
	K=K1+I		MTRS0495
	Q=(H(K)-2.*US(K)+H(J))		MTRS0496
	IF (Q)	29,30,29	MTRS0497
29	H(K)=H(J)- ((US(K)-H(J))*2 / Q)		MTRS0498
30	CONTINUE		MTRS0499
	NAKSR(MODE)=NAKSR(MODE) + 1		MTRS0500
	GOTO 17		MTRS0501
C	CONVERGENCE TEST NOT MET. RESTORE AND TRY AGAIN.		MTRS0503
32	DO 33 L=1,NC		MTRS0504
	J1=(L-1)*MAXR		MTRS0505
	DO 33 I=1,N		MTRS0506
	J=K1+J1+I		MTRS0507
	K=K5+J1+I		MTRS0508
	H(K)=US(J)		MTRS0509
33	US(J)=H(J)		MTRS0510
	NAK=2		MTRS0511
	GOTO 18		MTRS0512
C	IF PROBLEM COMPLEX, REPEAT ALL ABOVE FOR COMPLEX ARITHMETIC.		MTRS0514
36	DO 38 I=1,N		MTRS0515
	J=K5+I		MTRS0516
	K=K1+I		MTRS0517
	JJ=J*MAXR		MTRS0518
	KK=K+MAXR		MTRS0519
	Q = (US(K)-H(J))*2 + (US(KK)-H(JJ))*2		MTRS0520
	IF (Q)	37,361,37	MTRS0521
361	IF ((H(K)-US(K))*2 + (H(KK) - US(KK))*2)	32,38,32	MTRS0522
37	IF (((H(K)-US(K))*2 + (H(KK) - US(KK))*2) / Q-AT)	38,38,32	MTRS0523
38	CONTINUE		MTRS0525
	DO 39 I=1,N		MTRS0527
	J=K5+I		MTRS0528
	JJ=J*MAXR		MTRS0529
	K=K1+I		MTRS0530

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      KK=K+MAXR
      Q = (H(K)-2.*US(K)+H(J))*2 + (H(KK)-2.*US(KK)+H(JJ))*2
      IF ( Q ) 35,39,35
35  X=H(K)
      H(K)= H(J) - ( ((US(K)-H(J))*2-(US(KK)-H(JJ))*2)*(H(K)-2,*
      1  US(K)+H(J))+(2.*(US(K)-H(J))*(US(KK)-H(JJ))*
      2  (H(KK)-2.*US(KK)+H(JJ))) ) / Q
      H(KK)=H(JJ)-(((2.*(US(K)-H(J))*(US(KK)-H(JJ)))+(X-2,*
      1  US(K)+H(J))-((US(K)-H(J))*2-(US(KK)-H(JJ))*2)
      2  *(H(KK)-2.*US(KK)+H(JJ))) / Q )
39  CONTINUE
      NAKSR(MODE) = NAKSR(MODE) + 1
      GOTO 17
40  DO 41 J=1,NC
      J1=MAXR*(J-1)
      DO 41 I=1,N
      K=K1+J1+I
      L=K5+J1+I
41  H(L)=H(K)
      GOTO(18,56),IGO
44  DO 45 J=1,NC
      J1=MAXR*(J-1)
      DO 45 I=1,N
      K=K1+J1+I
45  US(K)=H(K)
      GOTO 18
C  MODE DID NOT CONVERGE IN NORMAL ITERATION
46  MODE = MODE-1
      PRINT 94,MODE
      GO TO 80
56  DO 58 J=1,NC
      J1=MAXR*(J-1)+INDEX
      DO 58 I=1,N
      K=K1+MAXR*(J-1)+I
      US(K)=A(J1)
58  J1=J1+K4
      CALL SWEEPX (VECTOR,A,H,US,EIGVAL,MODE,N,MAXR,NC,INDEX,EPSP)
59  J1=(NC-1)*MAXR+INDEX
      GUESS(J1)=0.
      GUESS(INDEX)=0.
62  IF (NMODE-MODE) 70,70,15
108 PRINT 131
      MODE = MODE-1
      PRINT 132,MODE
      NMODE = MODE
      GO TO 70
70  IF ( NTAPE ) 71,80,71
71  CALL DPMLTX (UTRSV,NC,VECTOR,NC,US,N,N,MODE,MAXR,MAXR,MAXR)
      J = 1
      K = 1
      DO 72 I=1,MODE
      INDEX = 0
      CALL NPNRMX (US(J),US(J),N,H(K),INDEX,MAXR,NC)
      J = J+K4
72  K = K+NC
80  PRINT 95
      DO 86 I=1,MODE
      LITR=NITER(I)-NITRSP
      IF ( LITR ) 81,82,82
81  LITR=0
      GOTO 85
82  NITER(I)=NITRSP

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MTRS0531
 MTRS0532
 MTRS0533
 MTRS0534
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 MTRS0650

 MTRS0666
 MTRS0667
 MTRS0668
 MTRS0669
 MTRS0670
 MTRS0671
 MTRS0672

```

85 GOTO (83,84),NC
83 PRINT 97, ( 1,EIGVAL(I),NITER(I),NAKSR(I))
      GOTO 86
84 L=2*I-1
      PRINT 96, ( 1, EIGVAL(L),EIGVAL(L+1),NITER(I)
1,NAKSR(I))
86 CONTINUE
      IF ( MODE ) 92,92,88
88 PRINT 98
      L=MODE*NC
      CALL MPRINT (VECTOR,N,L,MAXR)
      IF ( NTAPE ) 92,92,90
90 PRINT 99
      PRINT 93, (H(I),I=1,L)
      CALL MPRINT (US,N,L,MAXR)
92 RETURN
93 FORMAT (1H ,6E18.8)
94 FORMAT (5H MODE,1I4,40H HAS NOT CONVERGED IN MAXIMUM ITERATIONS//)
95 FORMAT (1H17X, 6H MODE 15X, 11H EIGENVALUE 26X,
1 10HITERATIONS 11X, 7HAITKENS /)
96 FORMAT (1H 1I11, 2E19.8, 1I22, 1I19 )
97 FORMAT (1H 1I11, 9X, 1E20.8, 9X, 1I22, 1I19 )
98 FORMAT (1H0 / 1H0 46X, 14H EIGENVECTORS ///)
99 FORMAT (1H0 / 1H0 36H CHECK EIGENVALUES AND EIGENVECTORS )
131 FORMAT (48H1ERROR IN ITERATION SUBROUTINE...( OVERFLOW ))
132 FORMAT (25H CALCULATION TERMINATED.,1I6,19H MODES ARE CORRECT.)
      END
      END
      END

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MTRS0675
MTRS0676
MTRS0678
MTRS0679
MTRS0680
MTRS0681
MTRS0682
MTRS0683
MTRS0684
MTRS0685
MTRS0686

MTRS0691
MTRS0693
MTRS0694
MTRS0695
MTRS0697
MTRS0698
MTRS0699

MTRS0701
MTRS0702
MTRS0704
MTRS0704
MTRS0704